

# BEM research and new developments in Brazil

Ney Augusto Dumont  
Civil Engineering Department – PUC-Rio



**PONTIFÍCIA UNIVERSIDADE CATÓLICA  
DO RIO DE JANEIRO**



# BEM research and new developments in Brazil

a tiny part of

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DO RIO DE JANEIRO**





(A view from this tiny part of  
Brazil: my office room's window)

# Prominent BEM-researchers in Brazil

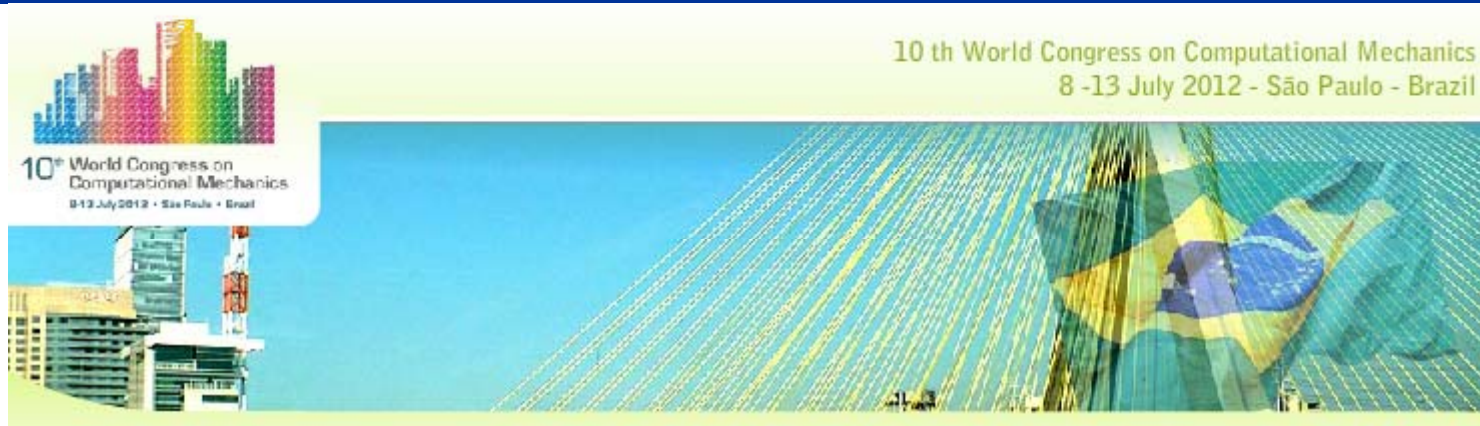
- José Cláudio Faria Telles
  - Webe João Mansur
  - Luiz Wrobel (at Brunel University)
- And large team at COPPE/UFRJ and other universities
- Euclides de Mesquita Neto
  - Paulo Sollero
- At UNICAMP
- Wilson Sérgio Venturini (passed away last July)
  - João Batista de Paiva
  - Humberto Breves Coda
- At São Carlos (UESC)
- Delfim Soares Júnior – at UFJF
  - Francisco Célio de Araújo – at UFOP
- New generation
-

# ANNOUNCEMENT

- **BETEQ 2011 - XII International Conference on Boundary Element and Meshless Techniques**  
**12-15 July 2011, Brasilia, Brazil**

A few days in Rio de Janeiro is mandatory!


# ANNOUNCEMENT



- Home
- Committee and Organising
- Social Program
- Technical Program
- Time Table
- Key Dates
- General Information
- Registration Information
- Abstracts
- Topics
- Hotel Accommodation
- Pre and Post Tours in Brazil
- Congress Secretariat
  
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## Congress Secretariat




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A few days in Rio de Janeiro is mandatory!



# Recently graduated student and works in progress at PUC-Rio

## Recently concluded Ph.D Thesis:

- M. F. F. Oliveira, *Conventional and simplified-hybrid boundary element methods applied to axisymmetric elasticity problems in fullspace and halfspace, PUC-Rio (2009). Co-advisor: Patrick Selvadurai (McGill University)*

## Ph.D Theses in progress

- C. A. Aguilar M., *Comparison of the computational performance of the advanced mode superposition technique with techniques that use numerical Laplace transforms (since September 2008).*
- D. Huamán M., *Gradient elasticity formulations with the hybrid boundary element method (since September 2008)*

## M.Sc. Thesis in progress

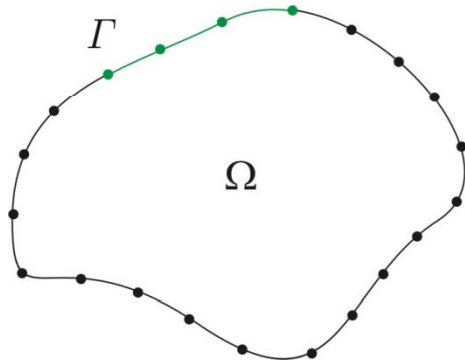
- E. Y. Mamani V., *Application of a generalized Westergaard stress function for the analysis of fracture mechanics problems (since January 2010). Co-advisor: A. A. O. Lopes*

# (Intended) Contents of this presentation

- • Variationally-based, hybrid boundary and finite elements
- Developments for time-dependent problems
- Developments in gradient elasticity
- Dislocation-based formulations (for fracture mechanics)
- From the collocation (conventional of hybrid) boundary element method to a meshless formulation  
Or: **The expedite boundary element method**



# Approximations on the boundary



$$\sigma_{ji,j} + b_i = 0$$



$$u_i = u_{im} d_m$$

$$t_i = t_{i\ell} t_\ell$$

$$\sigma_{ji,j}^p + b_i = 0$$

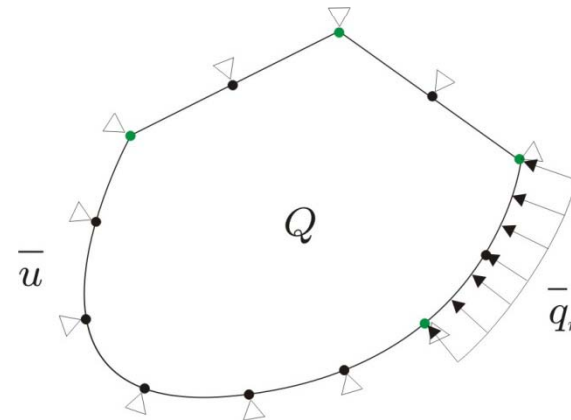


$$u_i^p = u_{im} d_m^p$$

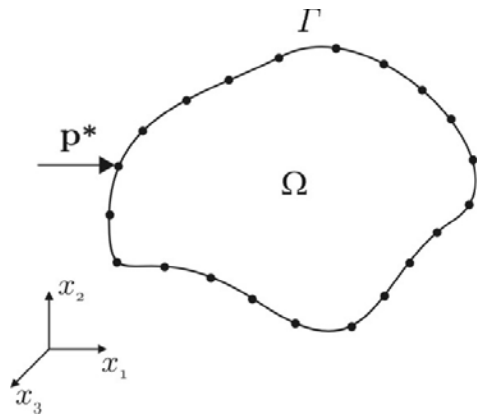
$$t_i^p = t_{i\ell} t_\ell^p$$

$d_m, d_m^p$  : nodal attributes

$t_\ell, t_\ell^p$  : surface attributes



# Fundamental solution



$$\sigma_{ji,j}^* + \Delta_{im} p_m^* = 0$$

$$u_i^* = u_{im}^* p_m^* + c_i^r \equiv u_{im}^* p_m^* + u_{is}^r C_{sm} p_m^* \quad \text{in } \Omega$$

$$\sigma_{ij}^* = \sigma_{ijm}^* p_m^* = D_{ijkp} u_{km,p}^* p_m^* \quad \text{in } \Omega$$

$$t_i^* = t_{im}^* p_m^* = \sigma_{jim}^* \eta_j p_m^* \quad \text{on } \Gamma$$

- Properties

$$\int_{\Omega} \sigma_{jim,j}^* d\Omega = -\delta_{im}$$

$$\int_{\Gamma} t_{im}^* d\Gamma = -\delta_{im}$$

# The conventional, collocation BEM

$$\mathbf{H}(\mathbf{d} - \mathbf{d}^p) \cong \mathbf{G}(\mathbf{t} - \mathbf{t}^p)$$

Derived from the Somigliana's identity

$\cong$  means congruence in terms of weighted residuals (collocation method)  
(inherent approximation error due to rigid body displacements!)

$\mathbf{d}^p, \mathbf{t}^p$  are already an expedite means of taking a particular solution into account

or

$$\left( \int_{\Gamma} \sigma_{jim}^* \eta_j u_{in} d\Gamma \right) d_n \equiv H_{mn} d_n \cong G_{ml} t_l \equiv \left( \int_{\Gamma} u_{im}^* t_{il} d\Gamma \right) t_l$$

$d_n$  = nodal displacement attributes

Where:  $u_{in}$  = displacement interpolation functions

$t_{il}$  = boundary traction force attributes

$t_{il}$  = traction force interpolation functions

$\sigma_{jim}^*, u_{im}^*$  Stress and displacement expressions of the problem's fundamental solution

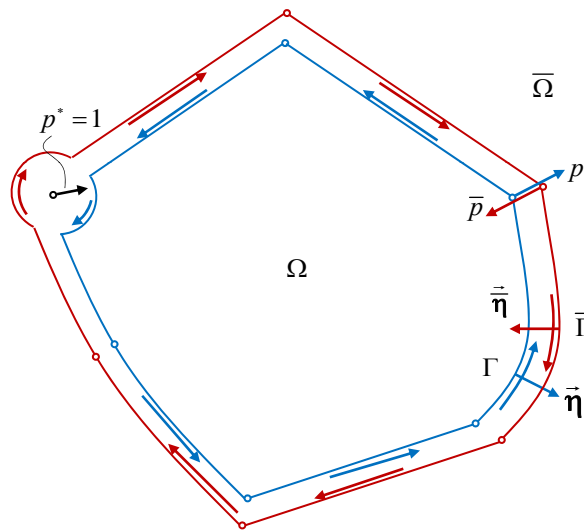
# Linear algebra properties of $\mathbf{H}$

$$\mathbf{d}^* = \mathbf{H}\mathbf{d} \iff \mathbf{p} = \mathbf{H}^T \mathbf{p}^*; \quad \bar{\mathbf{d}}^* = \bar{\mathbf{H}}\mathbf{d} \iff \bar{\mathbf{p}} = \bar{\mathbf{H}}^T \mathbf{p}^*$$

$$\mathbf{H} + \bar{\mathbf{H}} = \mathbf{I}$$

$$\mathbf{p} + \bar{\mathbf{p}} = \mathbf{p}^*$$

$$\mathbf{d}^* + \bar{\mathbf{d}}^* = \mathbf{d}$$



$$\mathbf{H}\mathbf{W} = \mathbf{0}; \quad \mathbf{H}^T \mathbf{V} = \mathbf{0}$$

$$\bar{\mathbf{H}}\mathbf{W} = \mathbf{W}; \quad \bar{\mathbf{H}}^T \mathbf{V} = \mathbf{V}$$

Spectral properties:

# Consistent formulation of the BEM

$$\mathbf{H}(\mathbf{d} - \mathbf{d}^p) = \mathbf{G}\mathbf{P}_R^\perp(\mathbf{t} - \mathbf{t}^p)$$

Where the orthogonal projector

$$\mathbf{P}_R^\perp = \mathbf{I} - \mathbf{P}_R = \mathbf{I} - \mathbf{R}(\mathbf{R}^T\mathbf{R})^{-1}\mathbf{R}^T$$

with

$$R_{ls} = \int_{\Gamma} t_{il} u_{is}^r d\Gamma$$

comes from the fact that there is an arbitrary amount of rigid body displacements in the fundamental solution

$$u_i^* = u_{im}^* p_m^* + c_i^r \equiv u_{im}^* p_m^* + u_{is}^r C_{sm} p_m^* \quad \text{in } \Omega$$

# Literature Review – part I

Hybrid formulations: { Coined by Pian in 1967 "... to signify elements which maintain either equilibrium or compatibility in the element and then to satisfy compatibility or equilibrium respectively along the interelement boundary"

## Parallel developments

Hellinger, E., 1914. Die allgemeinen Ansätze der Mechanik der Kontinua, *Enz. math. Wis*, **4**, 602-694.

Reissner, E. 1950. On a variational theorem in elasticity, *J. Math. Phys.*, **29**, 90-95 .

Hu, H.-C., 1955. On some variational principles in the theory of elasticity and the theory of plasticity, *Scientia Sinica*, **4**, 33-54

**Pian, T. H. H., 1964. Derivation of element stiffness matrices by assumed stress distribution. *AIAA J.*, **2**, 1333-1336**

Dumont, N. A., 1989. The hybrid boundary element method: an alliance between mechanical consistency and simplicity. *Applied Mechanics Reviews*, **42**, no. 11, Part 2, S54-S63

(and variations)

Trefftz, E., 1926. Ein Gegenstück zum Ritzschen Verfahren. *Proc. 2nd International Congress of Applied Mechanics*, Zurich, Switzerland.

Jirousek, J. & Leon, N., 1977. A powerful finite element for plate bending, *Com. Meths. in Appl. Mech. Engng.*, **12**, 77-96.

Qin, Q. H., 2003. *The Trefftz Finite and Boundary Element Method*, WITPress.

(Many variations)

# Present theoretical investigation

Motivation: we are here

Double layer potential matrix

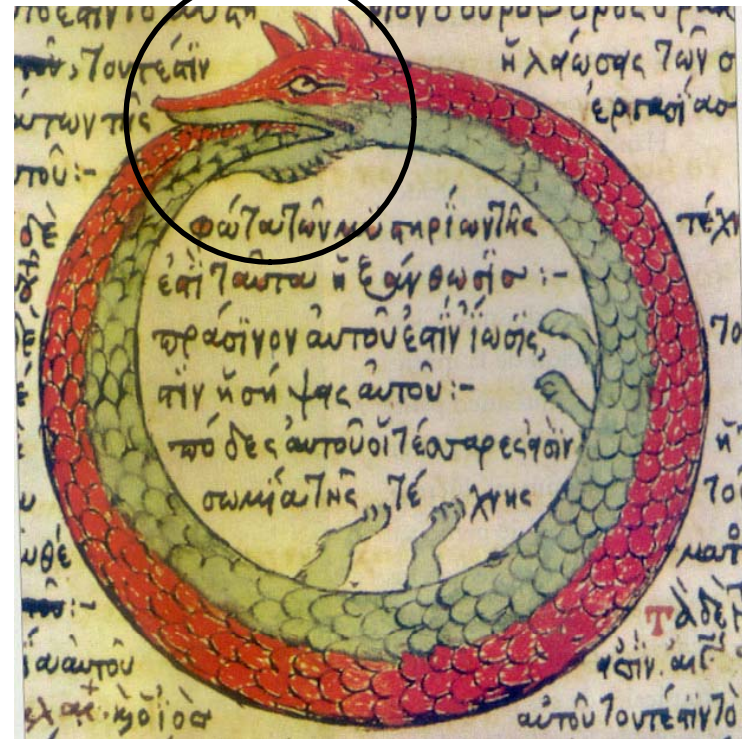
$$H_{mn} = \int_{\Gamma} \sigma_{jim}^* \eta_j u_{in} d\Gamma$$

Mechanical assumptions,  
linear algebra consequences

Theoretical tool:

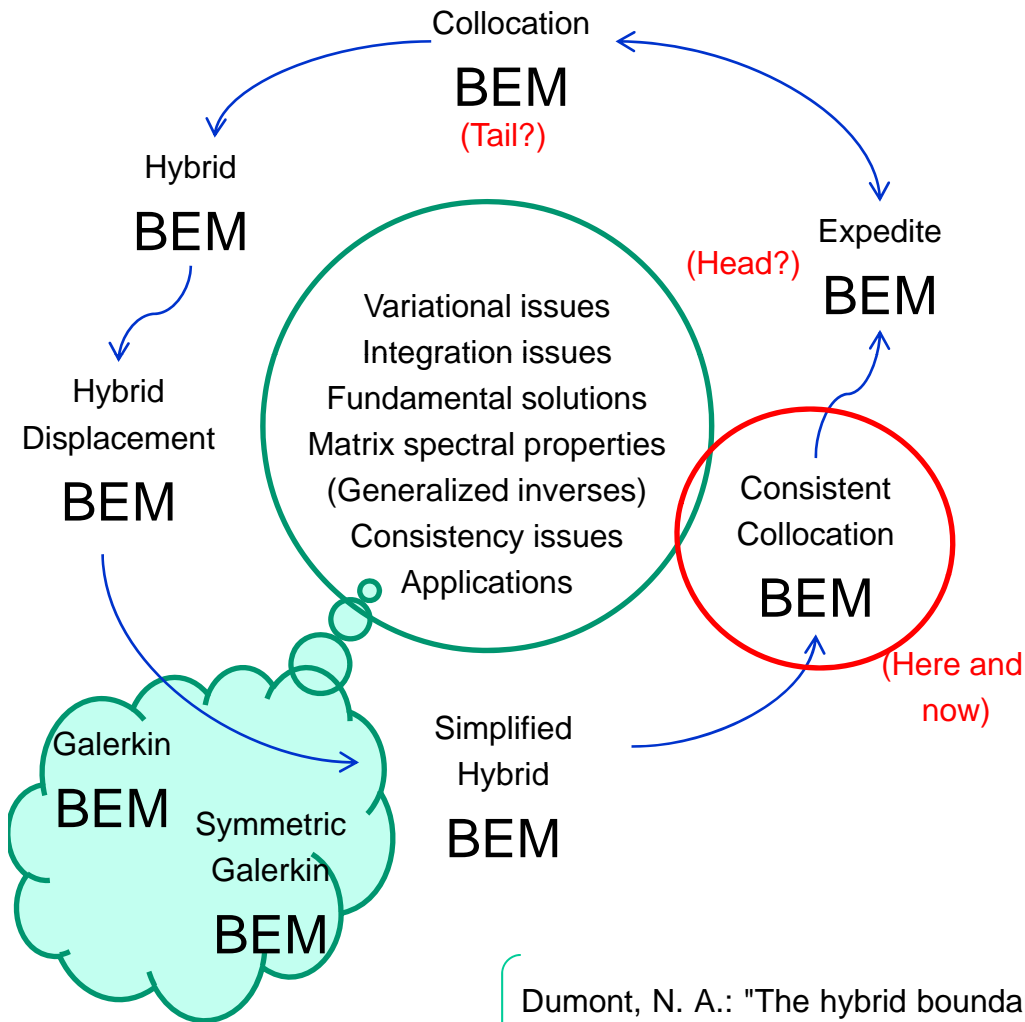
Displacement virtual work  
principle (elastostatics):

$$\mathbf{d}^* = \mathbf{H}\mathbf{d} \quad \Leftrightarrow \quad \mathbf{p} = \mathbf{H}^T \mathbf{p}^*$$



The Ouroboros (from Wikipedia)

# The Ouroboros!



(from Wikipedia)

Dumont, N. A.: "The hybrid boundary element method – fundamentals", in preparation to be submitted to Engineering Analysis with Boundary Elements



# The conventional, collocation BEM

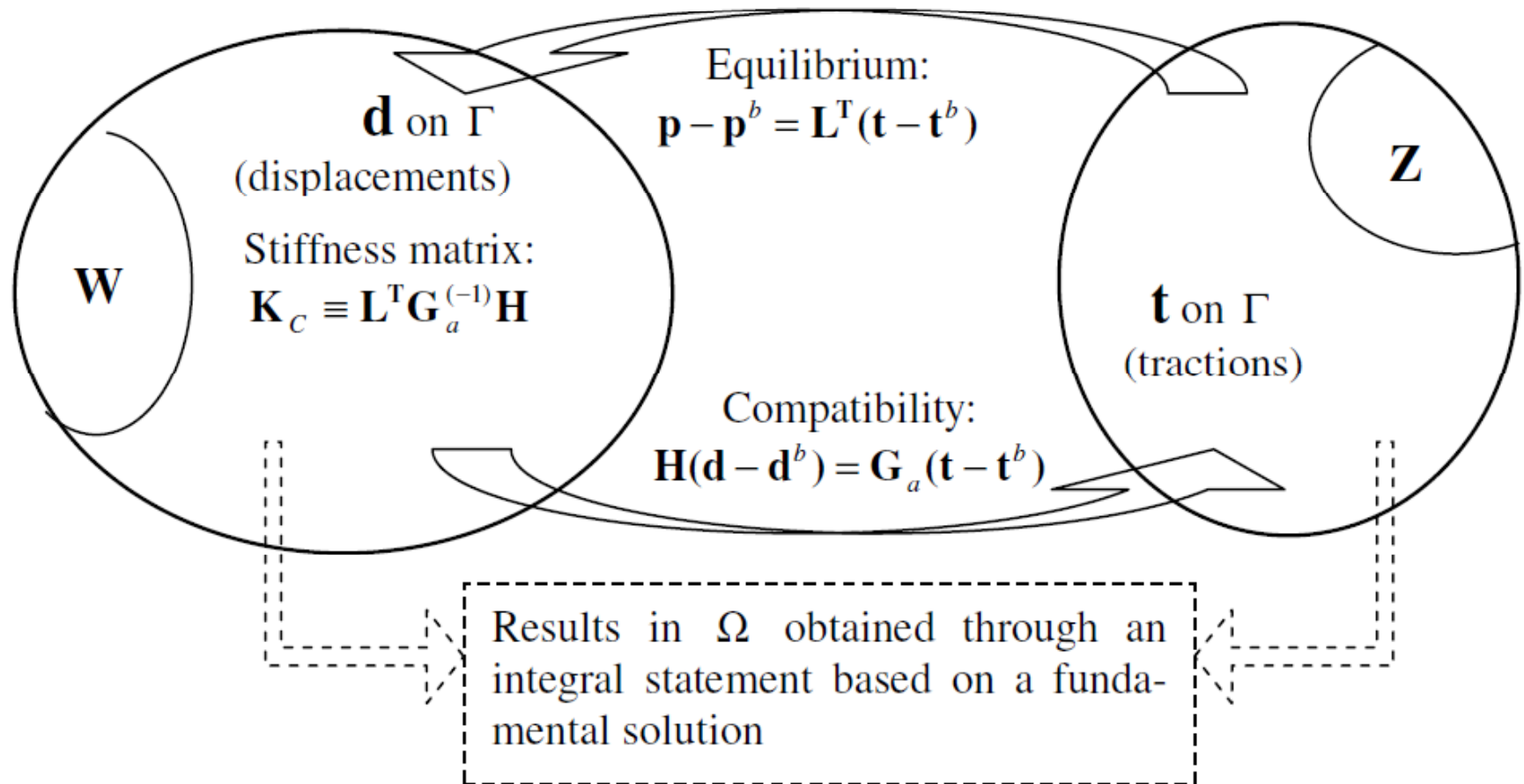


Figure 5. Transformations carried out in the conventional boundary element method.

# The hybrid BEM

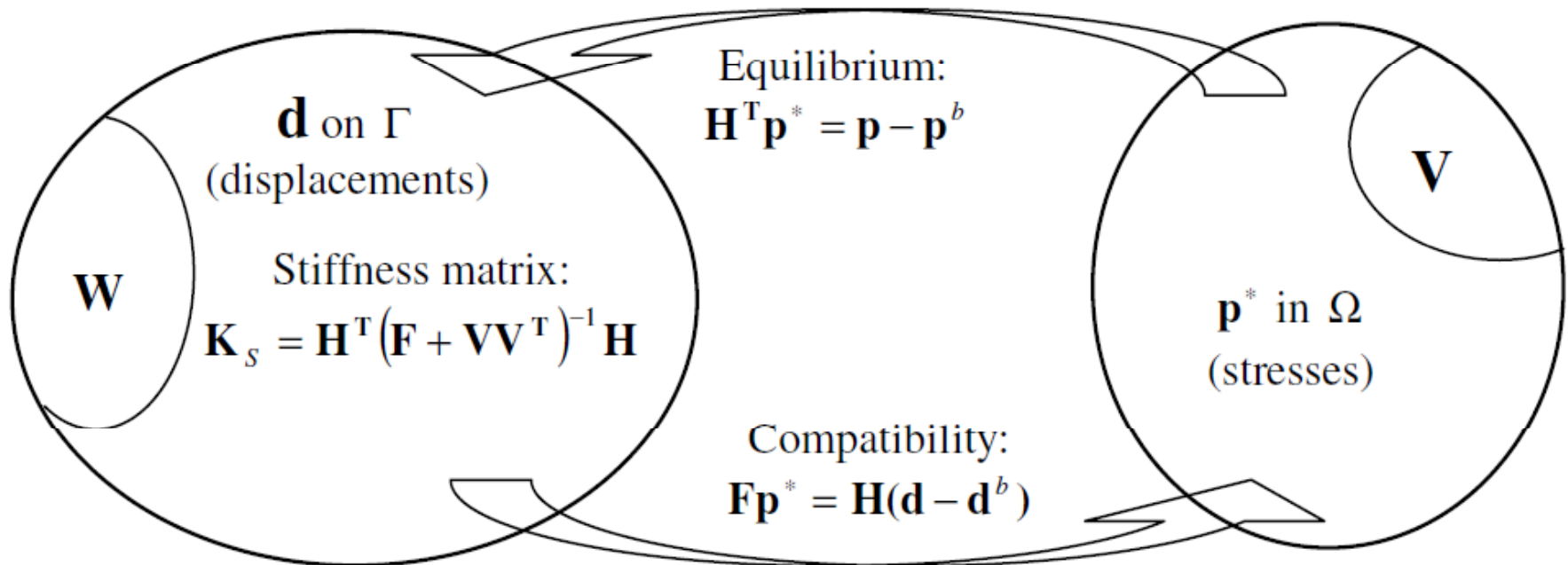


Figure 7. Transformations carried out in the hybrid stress boundary element method.

Dumont, N. A.: "Variationally-Based, Hybrid Boundary Element Methods",  
Computer Assisted Mechanics and Engineering Sciences (CAMES) Vol 10 pp 407-430, 2003

# The hybrid displacement BEM

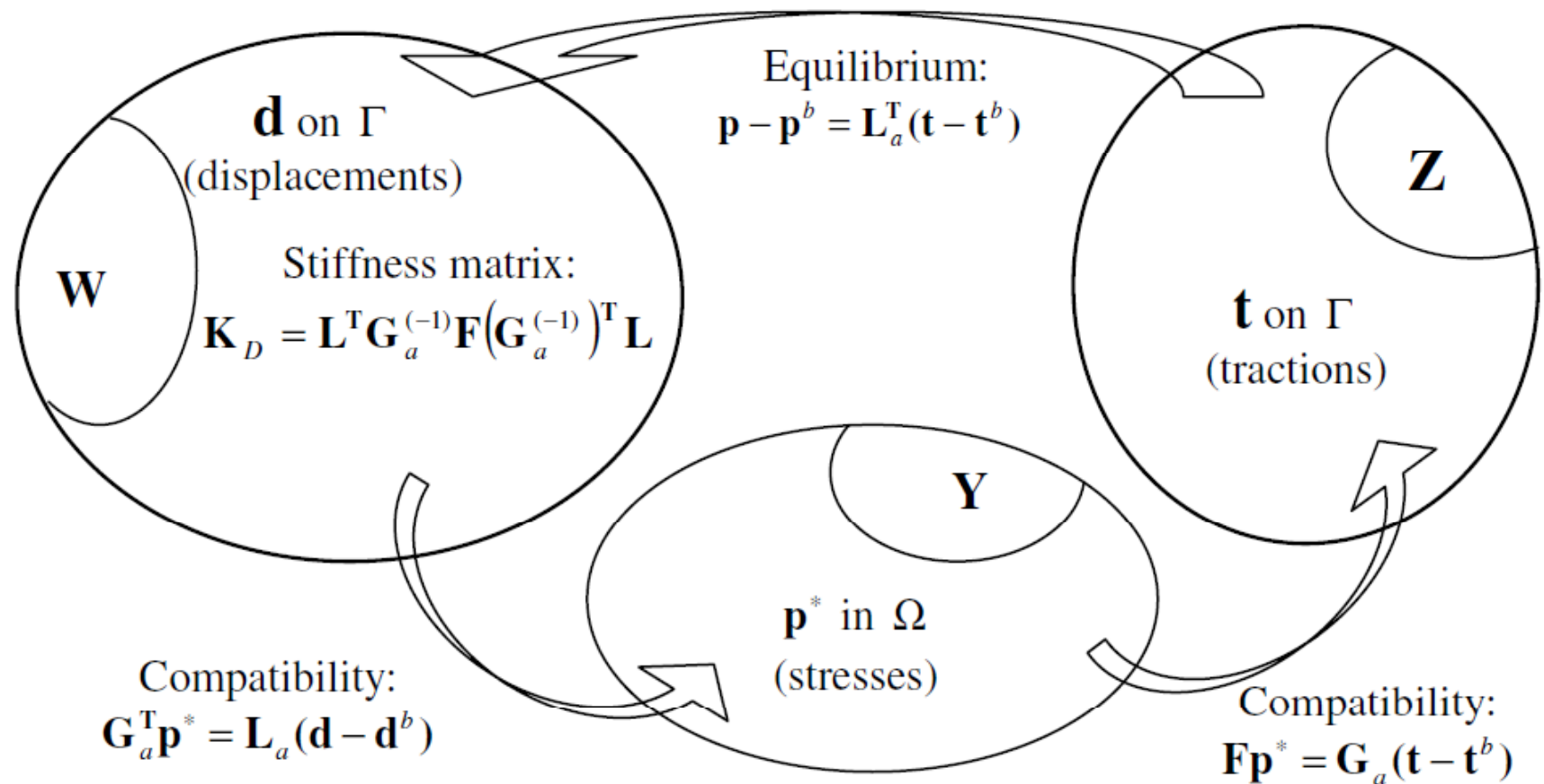


Figure 6. Transformations carried out in the hybrid displacement boundary element method.

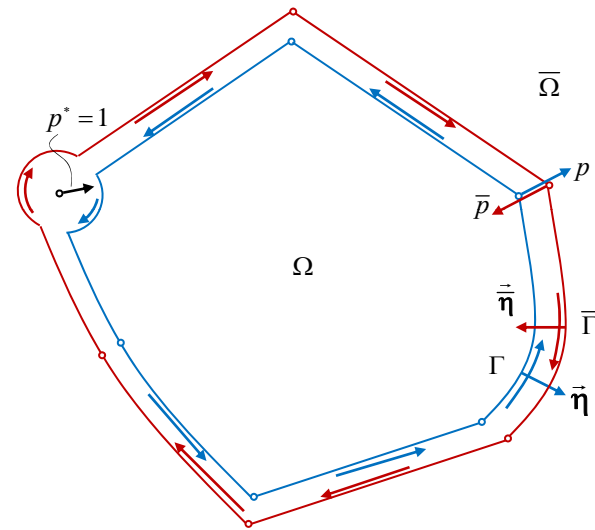
# Definition of the matrices involved

$$H_{mn} := \int_{\Gamma} \sigma_{jim}^* \eta_j u_{in}^* d\Gamma$$

$$G_{ml} := \int_{\Gamma} u_{im}^* t_{il} d\Gamma$$

$$F_{mn}^* := \int_{\Gamma} \sigma_{jim}^* \eta_j u_{in}^* d\Gamma$$

$$L_{lm} := \int_{\Gamma} u_{im}^* t_{il} d\Gamma$$



Simplified HBEM:  $\mathbf{F}^* \leftarrow \mathbf{H}\mathbf{U}^*$

# The Ouroboros!



Some isolated virtual work statements

$$\mathbf{H}^T \mathbf{p}^* = \mathbf{p} \quad \Leftrightarrow \quad \mathbf{H} \mathbf{d} = \mathbf{d}^*$$

$$\mathbf{G} \mathbf{t} = \mathbf{d}^* \quad \Leftrightarrow \quad \mathbf{G}^T \mathbf{p}^* = \mathbf{d}^L$$

$$\mathbf{L}^T \mathbf{t} = \mathbf{p} \quad \Leftrightarrow \quad \mathbf{L} \mathbf{d} = \mathbf{d}^L$$

There are more virtual work statements;  
Several restrictions apply!

$(\mathbf{d}, \mathbf{p})$  External reference system

$(\mathbf{p}^*, \mathbf{d}^*)$  Internal reference system

$(\mathbf{t}, \mathbf{d}^L)$  System of Lagrange multipliers

$\mathbf{G}$  and  $\mathbf{L}$  should be rectangular, in general;

$\mathbf{G}$  should be consistent

Dumont, N.A., An assessment of the spectral properties of the matrix  $\mathbf{G}$  used in the boundary element methods, *Computational Mechanics*, **22**(1), pp. 32-41, 1998

# The Ouroboros!



Some isolated virtual work statements

$$\mathbf{H}^T \mathbf{p}^* = \mathbf{p} \quad \Leftrightarrow \quad \mathbf{H} \mathbf{d} = \mathbf{d}^*$$

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$\mathbf{G}$  and  $\mathbf{L}$  should be rectangular, in general;

$\mathbf{G}$  should be consistent

$$\mathbf{F}^* \mathbf{p}^* = \mathbf{d}^*$$



$$\mathbf{K} \mathbf{d} = \mathbf{p}$$

Results at internal points: no integral statement actually required

# Spectral properties for the hybrid BEM

For a finite domain:

$\mathbf{P}_W$  is the orthogonal projector onto the space of rigid body displacements and  $\mathbf{P}_W^\perp = \mathbf{I} - \mathbf{P}_W$

Then,  $\mathbf{H}\mathbf{P}_W = \mathbf{0} \Rightarrow \mathbf{H}^T\mathbf{P}_V = \mathbf{0}$  ← Definition of  $\mathbf{P}_V$

For consistency,  $\mathbf{F}^*\mathbf{P}_V = \mathbf{0}$

and  $\mathbf{K} = \mathbf{H}^T \left( \mathbf{F}^* + \mathbf{P}_V \right)^{-1} \mathbf{H}$  in  $\mathbf{K}\mathbf{d} = \mathbf{p}$

Results at internal points: no integral statement actually required!

$\mathbf{p}^*$  solved from either  $\mathbf{F}^*\mathbf{p}^* = \mathbf{H}\mathbf{d}$  or  $\mathbf{H}^T\mathbf{p}^* = \mathbf{p}$  with  $\mathbf{P}_V\mathbf{p}^* = \mathbf{0}$

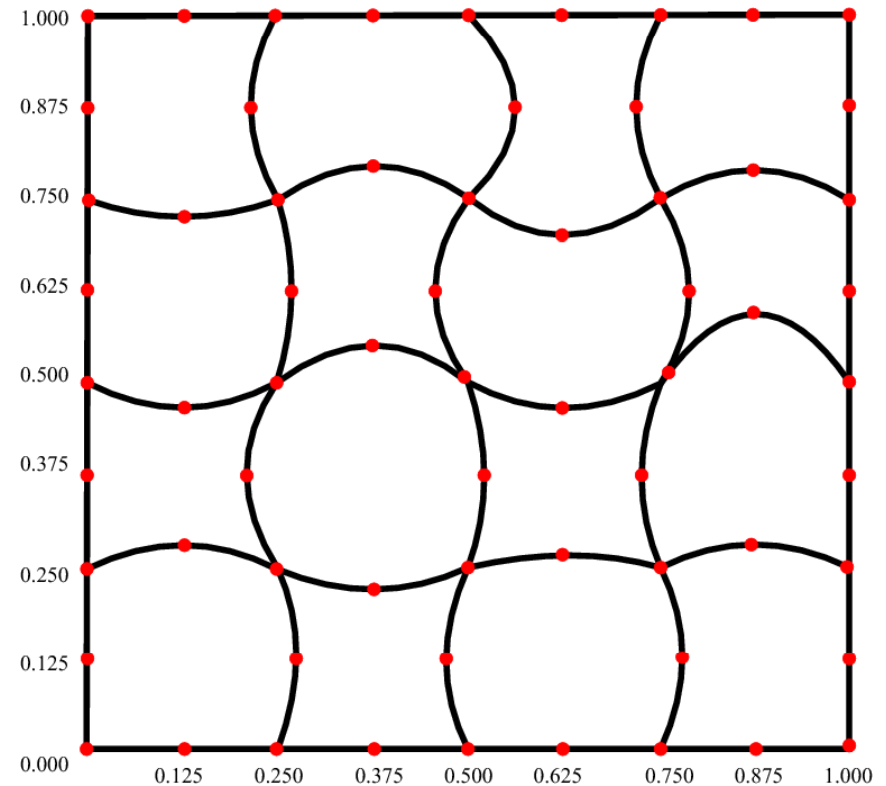
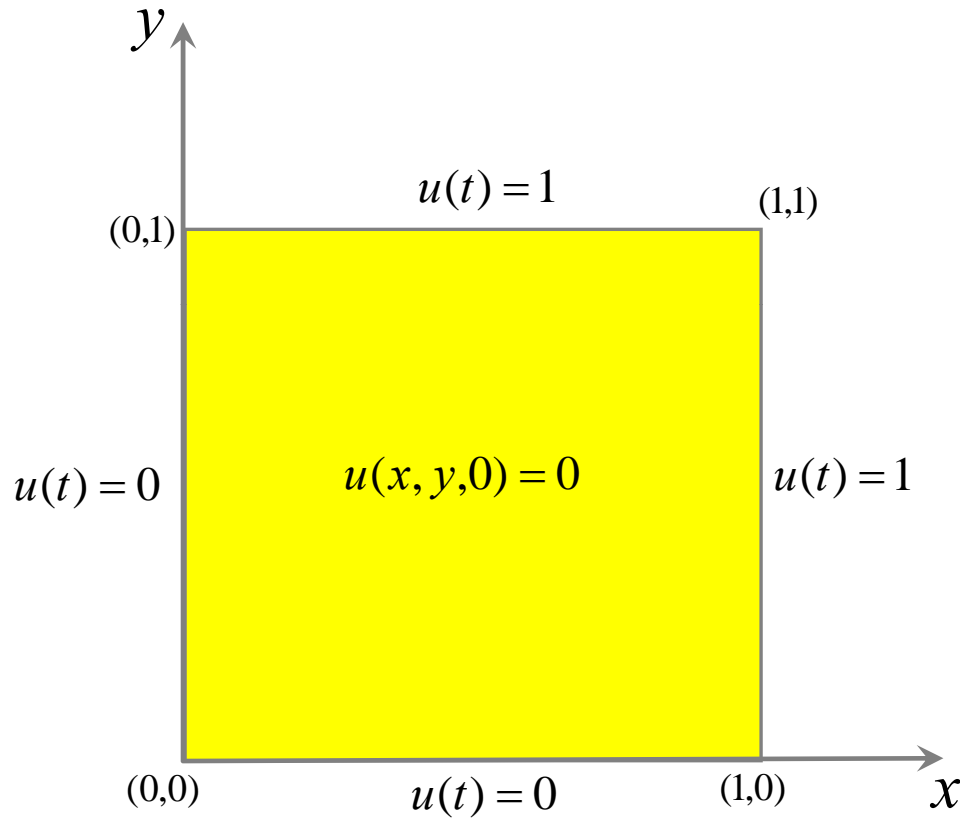
and  $u_i^* = u_{im}^* p_m^* + c_i^r$   $\sigma_{ij}^* = \sigma_{ijm}^* p_m^*$

# Some applications already implemented

- 2D and 3D potential and elasticity problems
- 2D and 3D acoustics
- 2D general time-dependent problems (in the frequency domain):
  - Advanced mode superposition analysis
  - Use of numerical inverse transforms
- 2D sensitivity analysis
- 2D FGM-analysis for potential problems
- 2D fracture-mechanics analysis – evaluation of stress intensity factors
- Axysymmetric fullspace and halfspace elastostatics
- 2D (and 3D) gradient elasticity analysis



# Example of application of non-singular fundamental solutions: two dimensional transient heat conduction in a homogeneous square plate

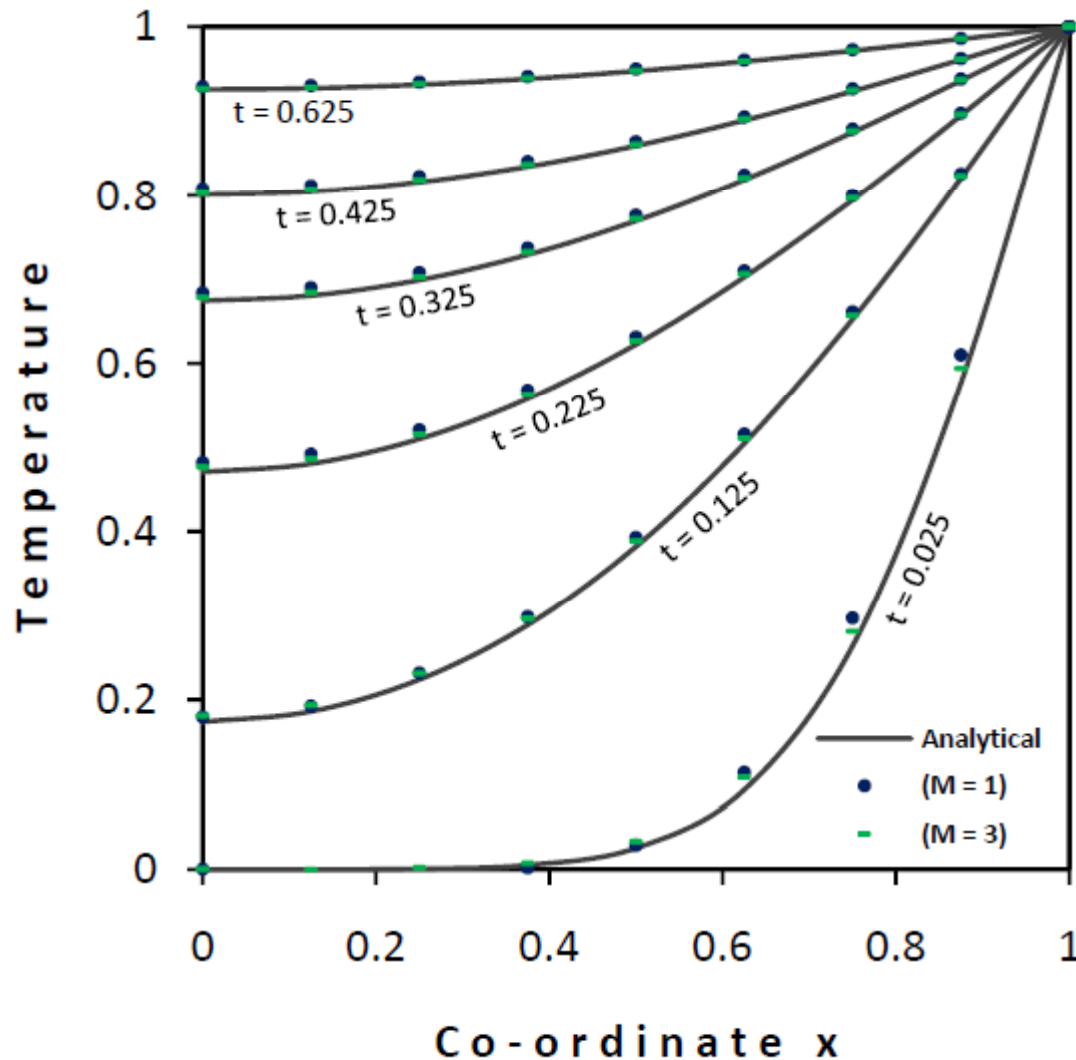


$k = 1$  Conductivity

$c = 1$  Specific heat

Distorted 4x4 mesh

# Temperature results along the edge $x = 0$ for several time instants



Analytic solution

Classical modal analysis.

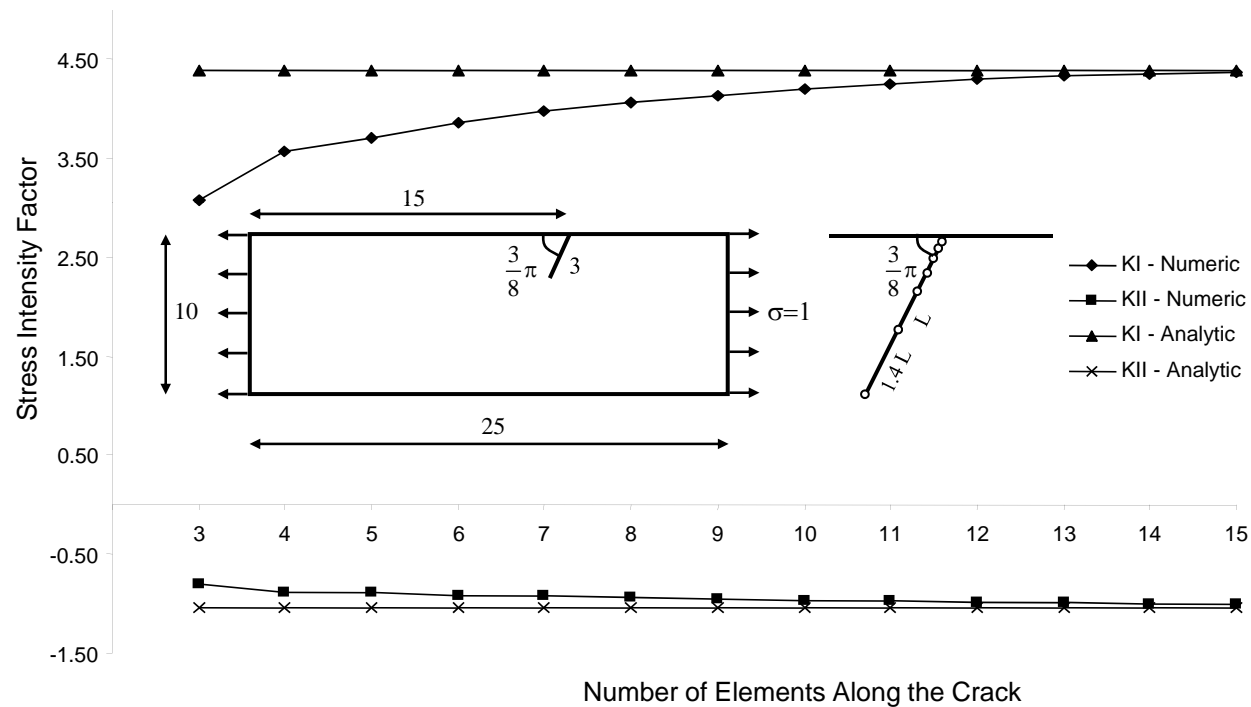
Serendipity elements.  
Quadratic elements.

Hybrid formulation

3 mass matrices.

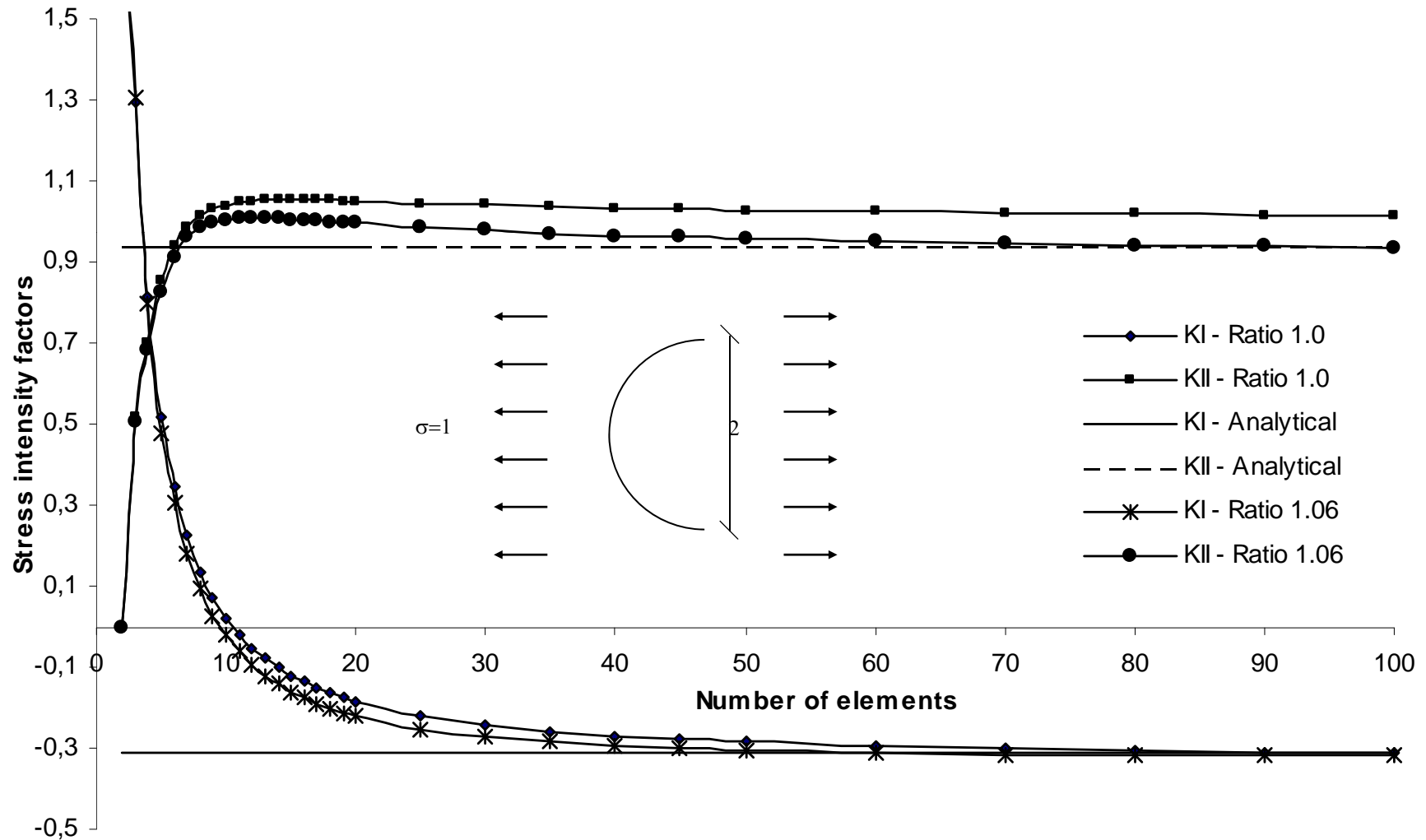
# Westergaard's Stress Function as a Fundamental Solution

## - Example 8 – inclined edge crack in a rectangular plate



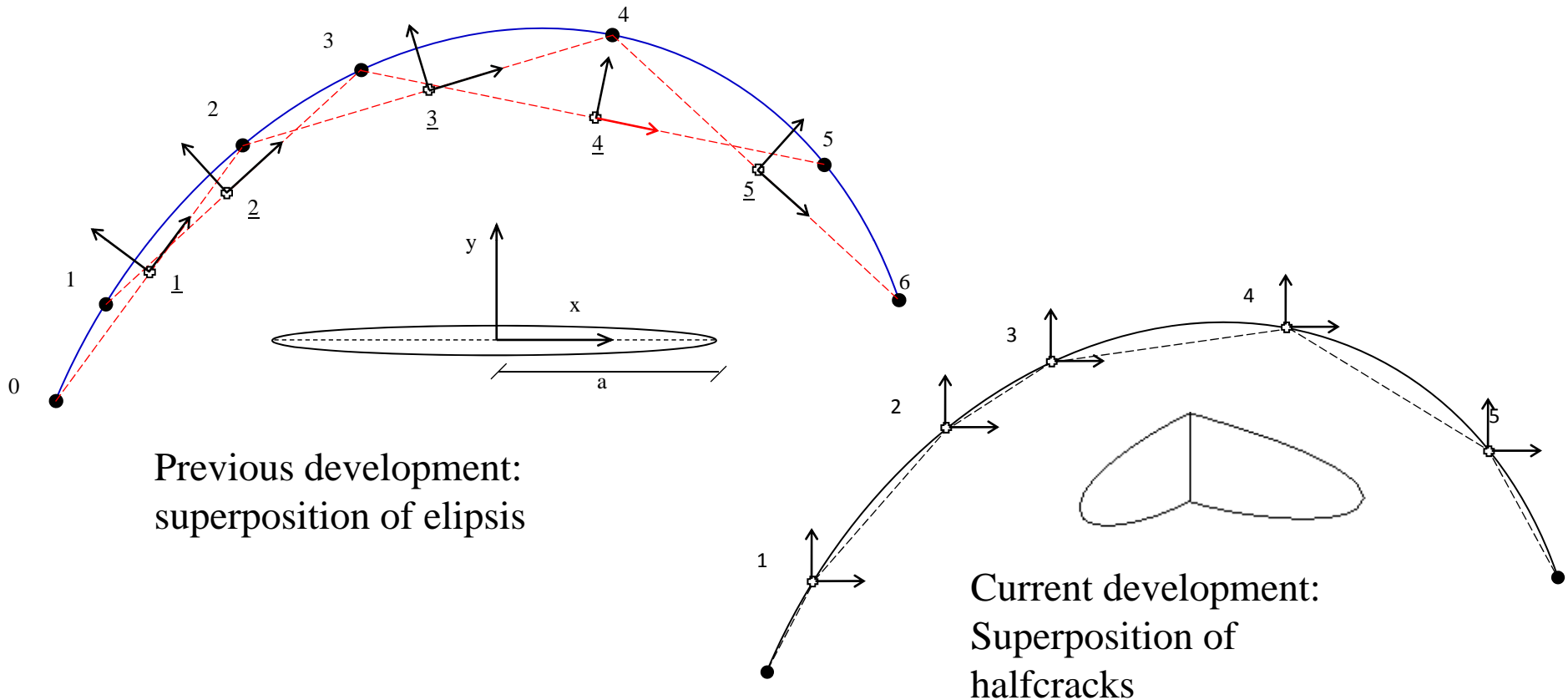
# Westergaard's Stress Function as a Fundamental Solution

## Example 3 – semicircular crack in an infinite continuum



# Westergaard's Stress Function as a Fundamental Solution

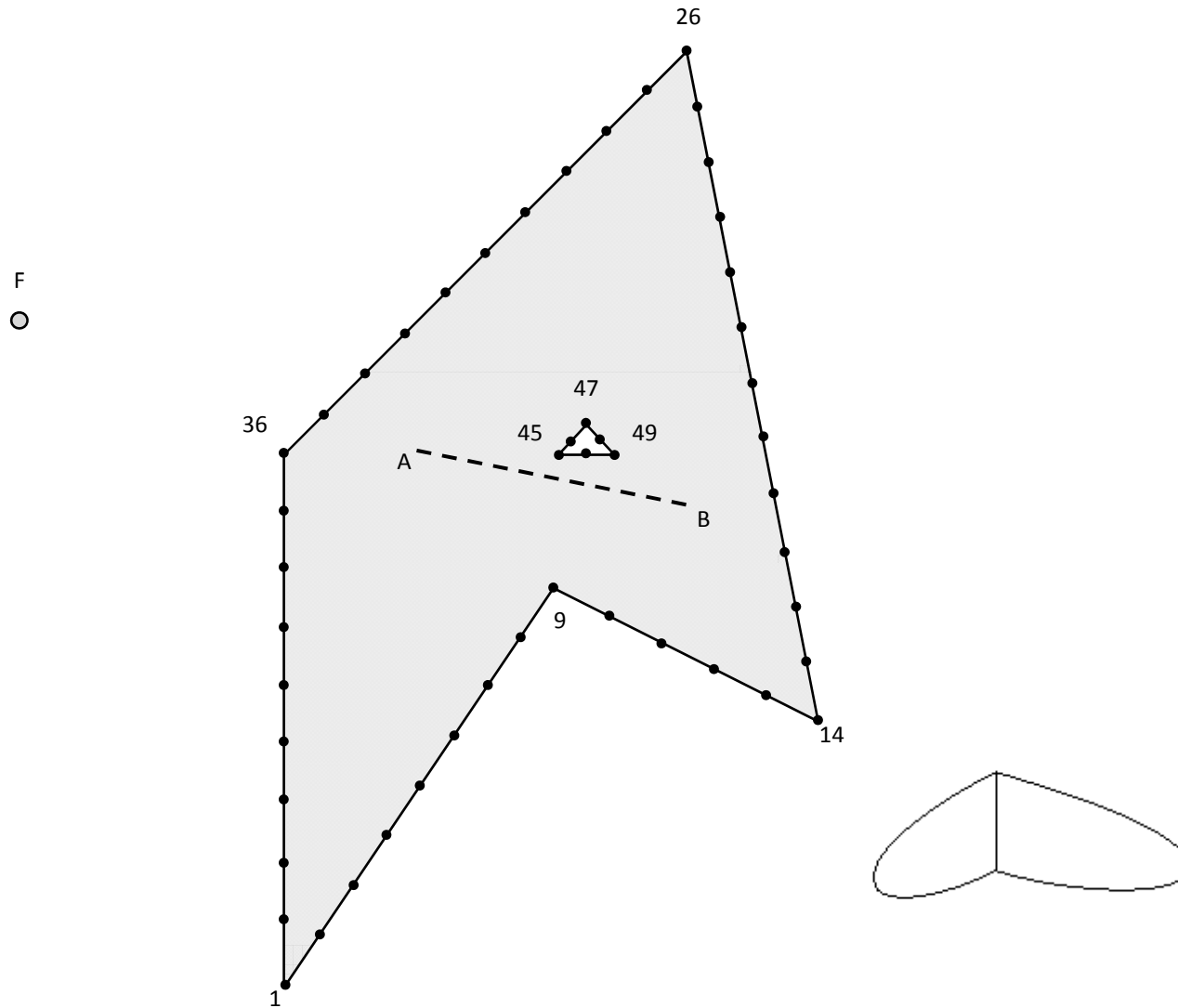
## Modeling a curved crack



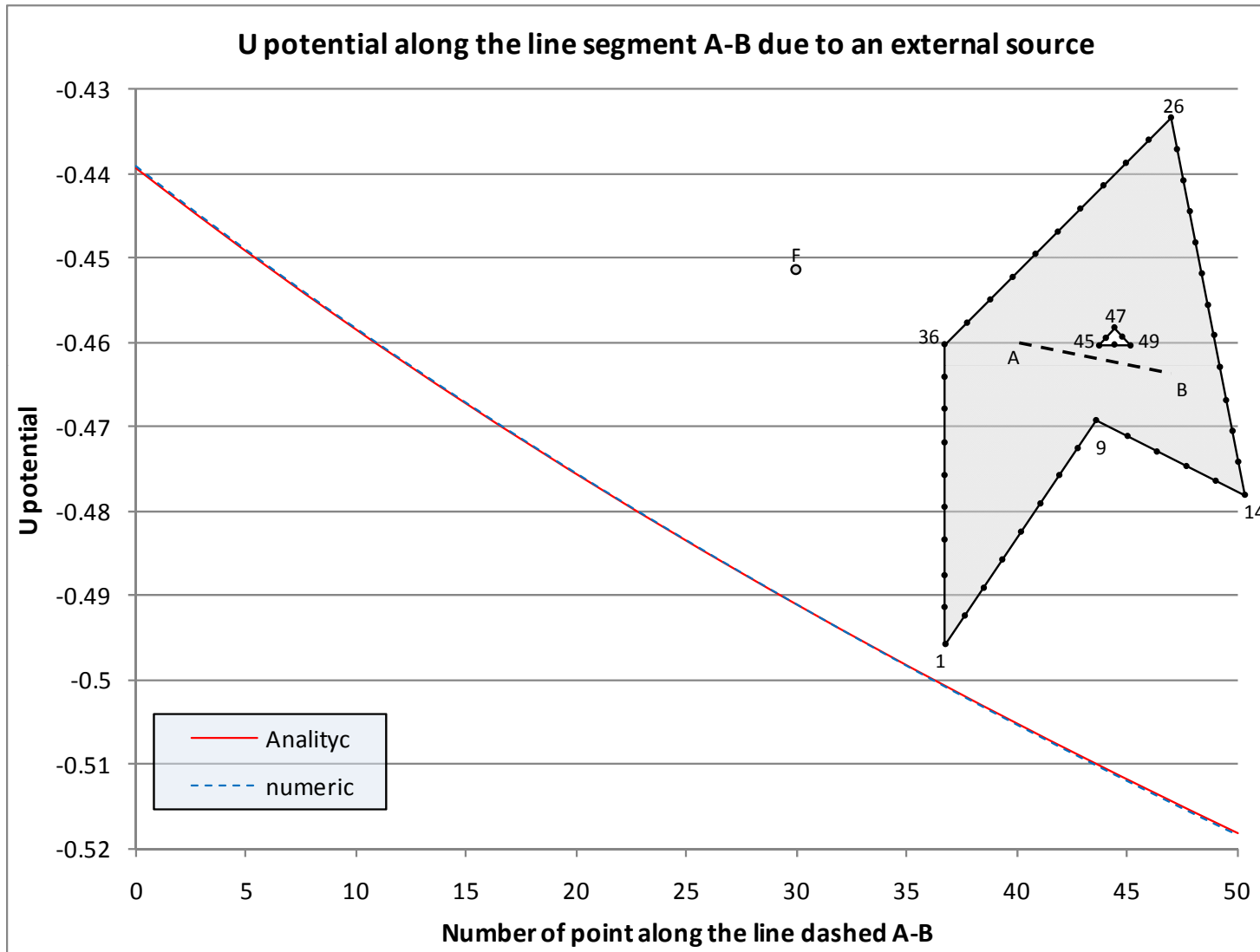
Inspired by:

Tada, H., Ernst, H. A., Paris, P. C. (1993), Westergaard stress functions for displacement-prescribed crack problems – I, Int. Journal of Fracture, Vol. 61, pp 39-53

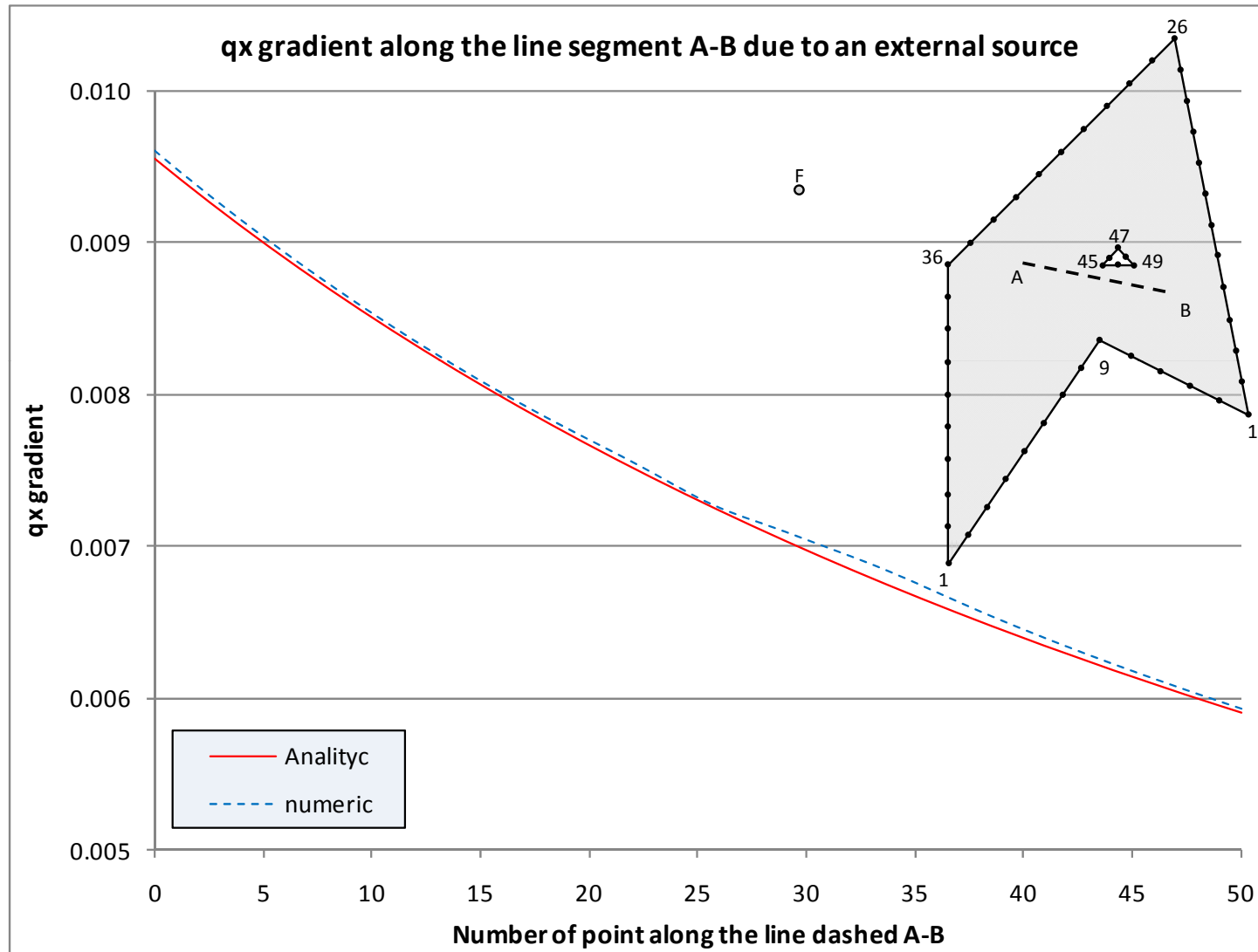
# Westergaard's Stress Function as a Fundamental Solution



# Westergaard's Stress Function as a Fundamental Solution

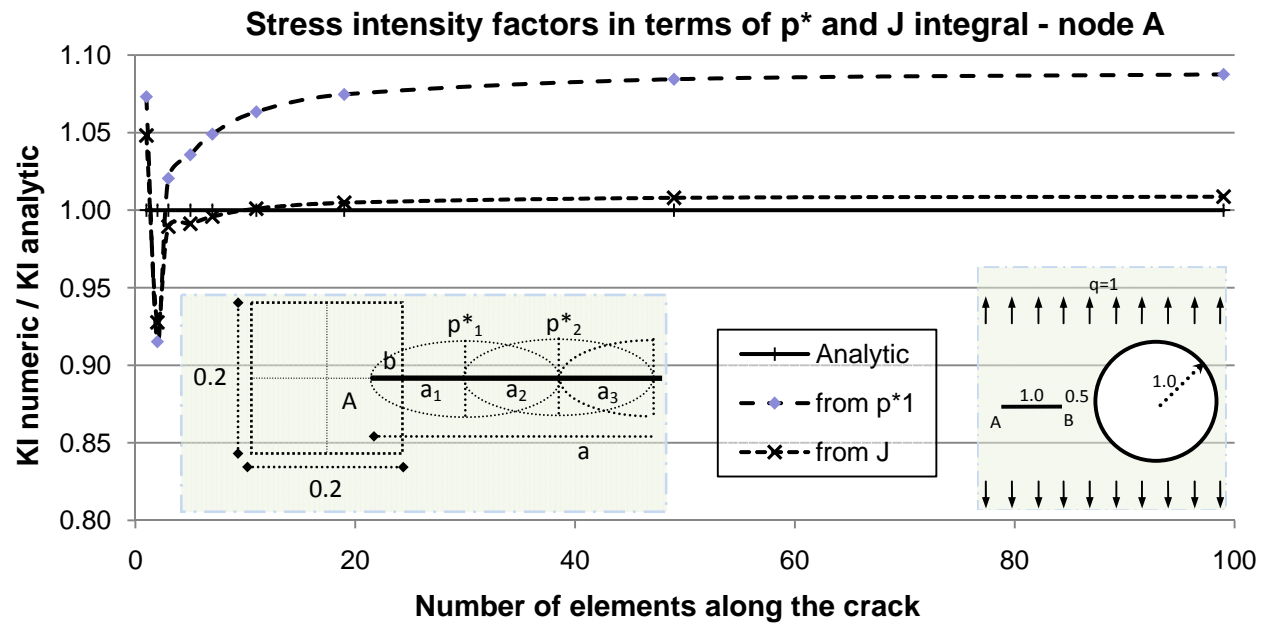


# Westergaard's Stress Function as a Fundamental Solution

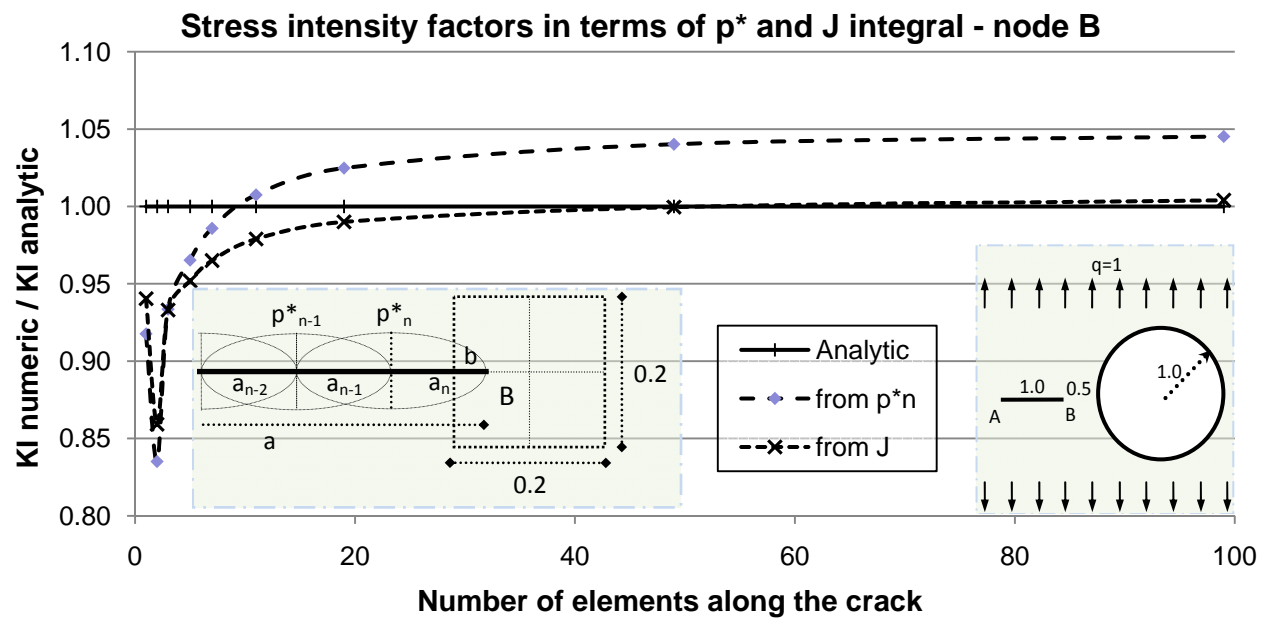




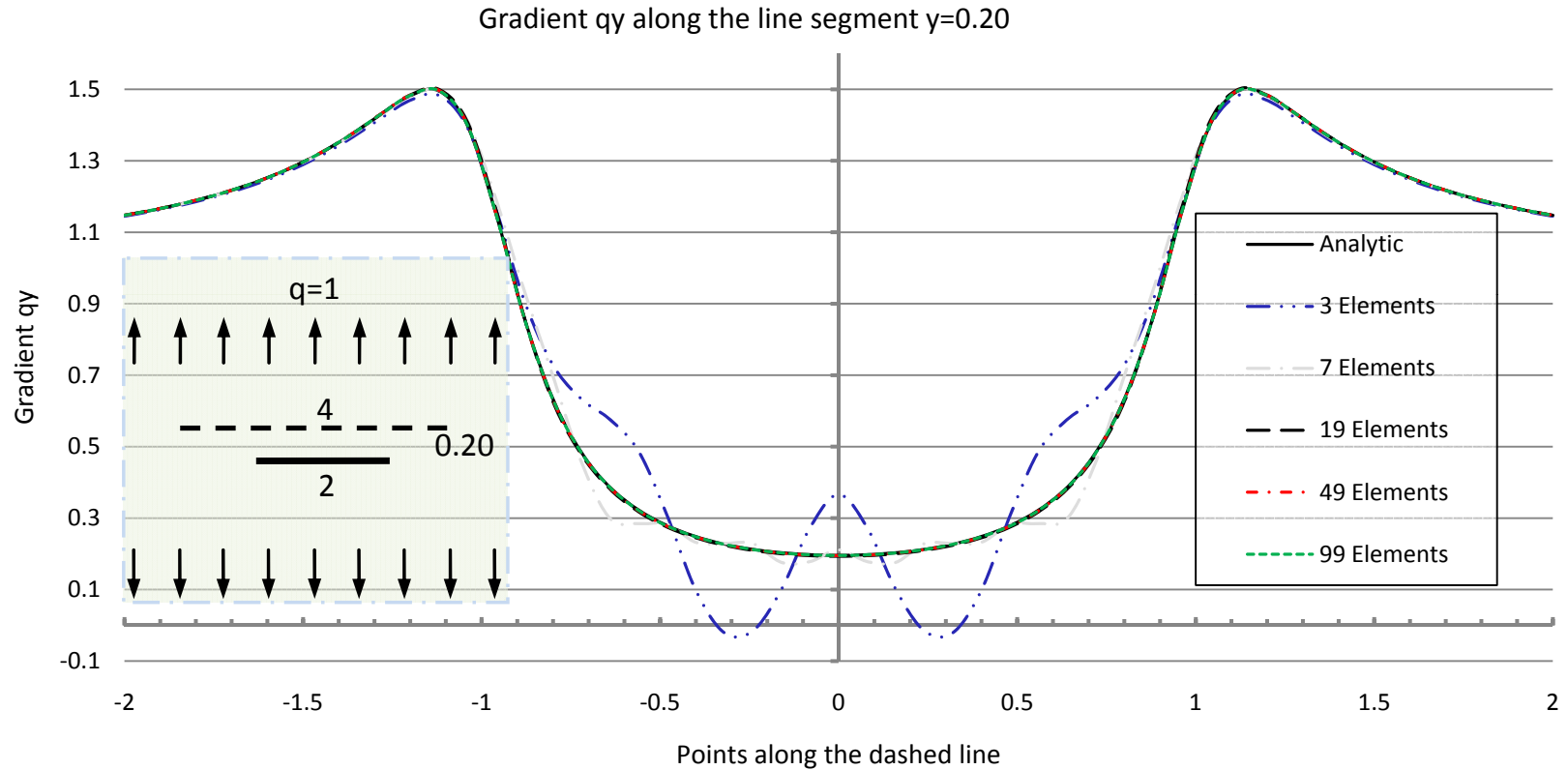
# Westergaard's Stress Function as a Fundamental Solution



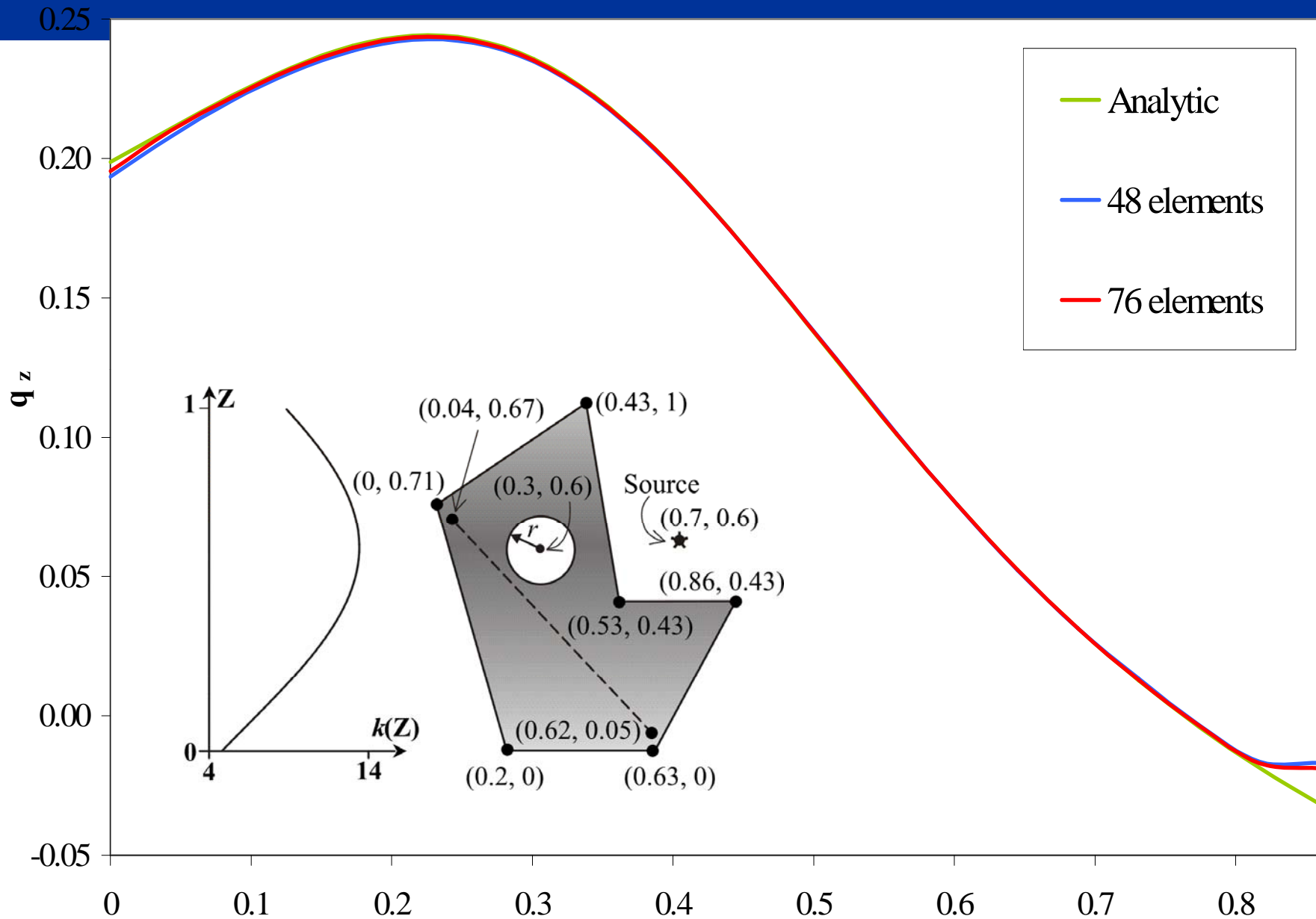
# Westergaard's Stress Function as a Fundamental Solution



# Westergaard's Stress Function as a Fundamental Solution

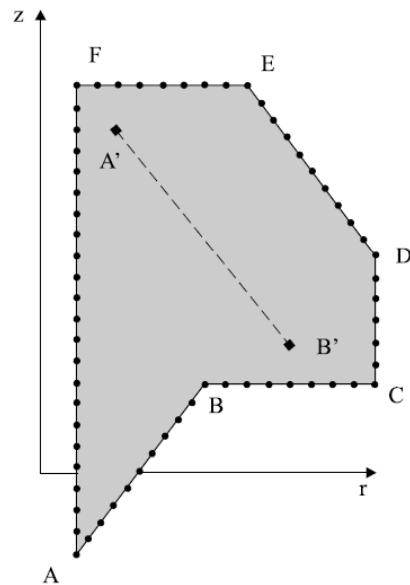


## EXAMPLE: gradient $q_z$ along a line segment

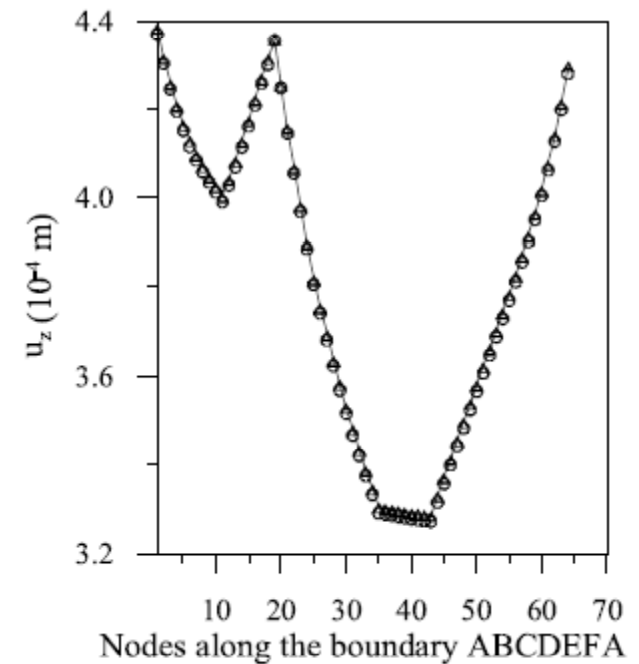
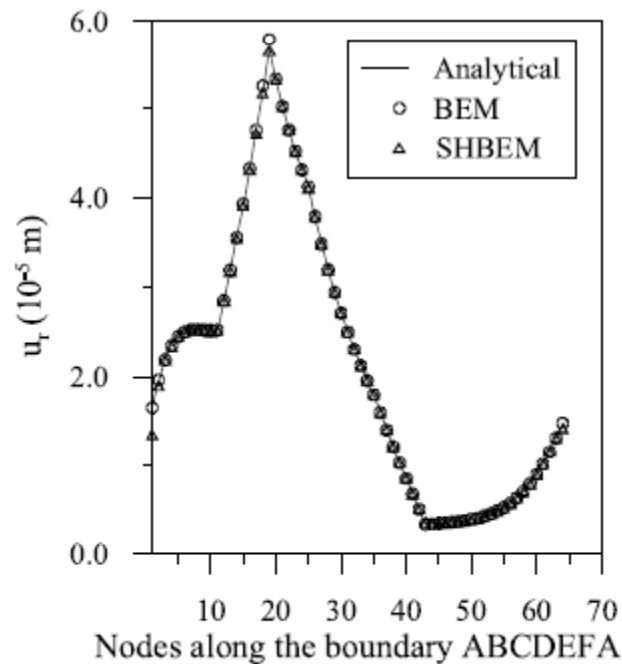


# Numerical example

Non-convex axisymmetric volume with multiply connected surfaces



|                 |                 |
|-----------------|-----------------|
| A = (1.0, -2.0) | A' = (2.0, 0.0) |
| B = (4.0, 2.0)  | B' = (5.0, 8.0) |
| C = (8.0, 2.0)  | C' = (2.0, 8.0) |
| D = (8.0, 5.0)  | D' = (6.0, 3.0) |
| E = (4.0, 9.0)  |                 |
| F = (1.0, 8.0)  |                 |

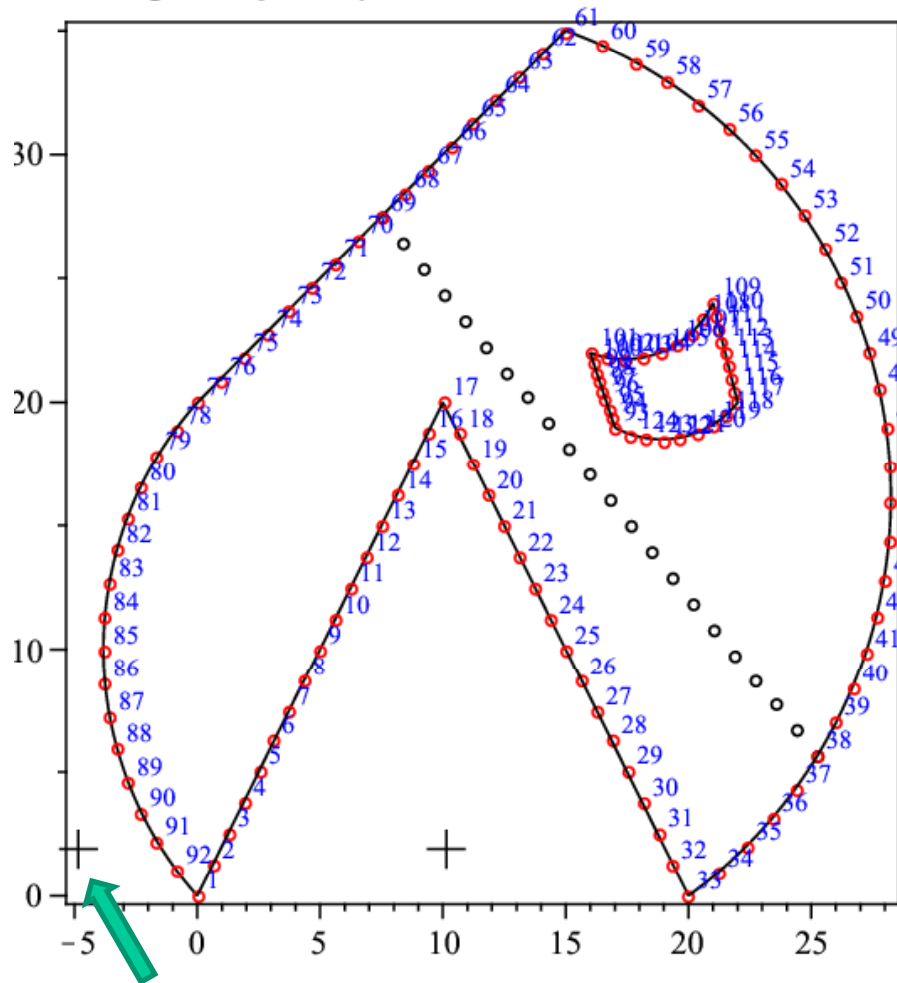


- Unitary ring sources (10, -5)

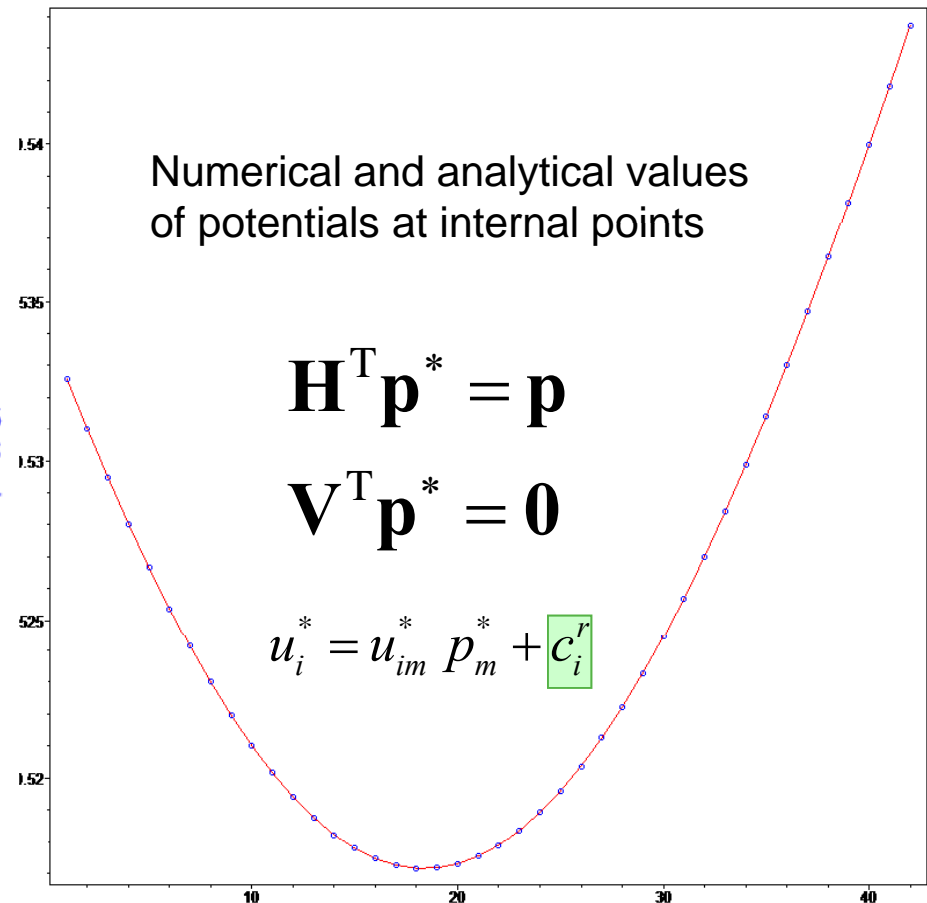
# An expedite formulation of the BEM

## Some numerical assessments for 2D potential problems

*Irregularly shaped structure with 124 nodes*



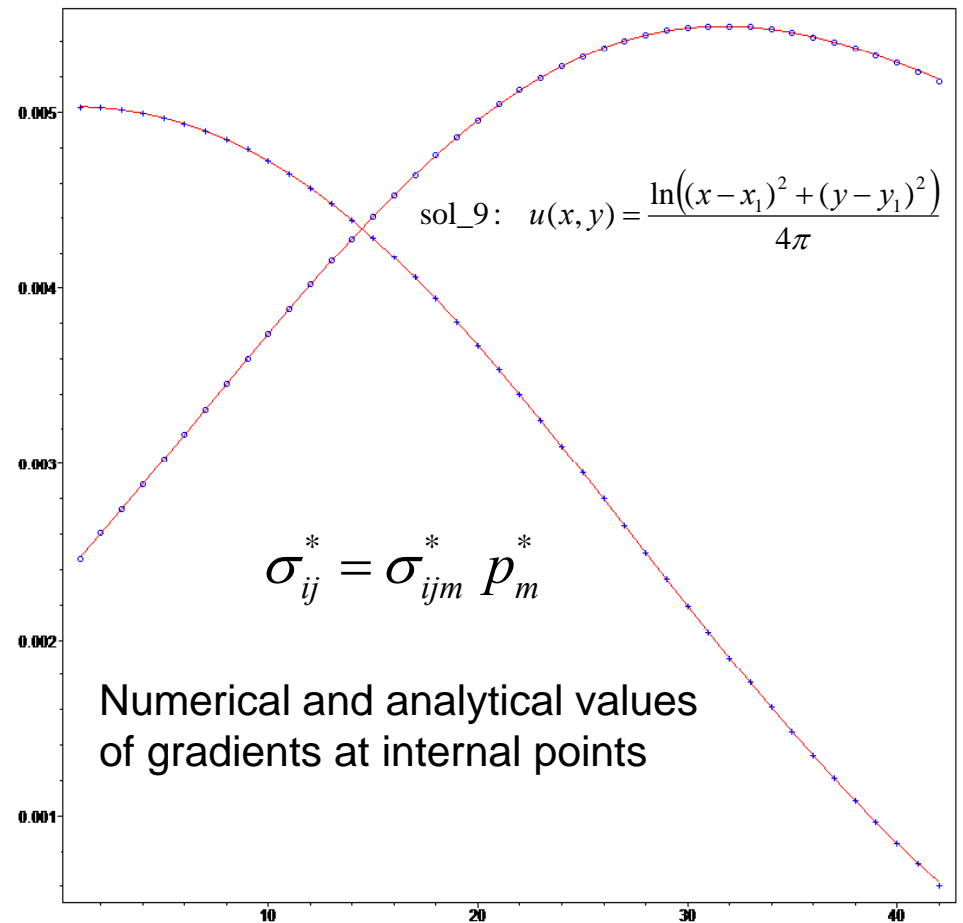
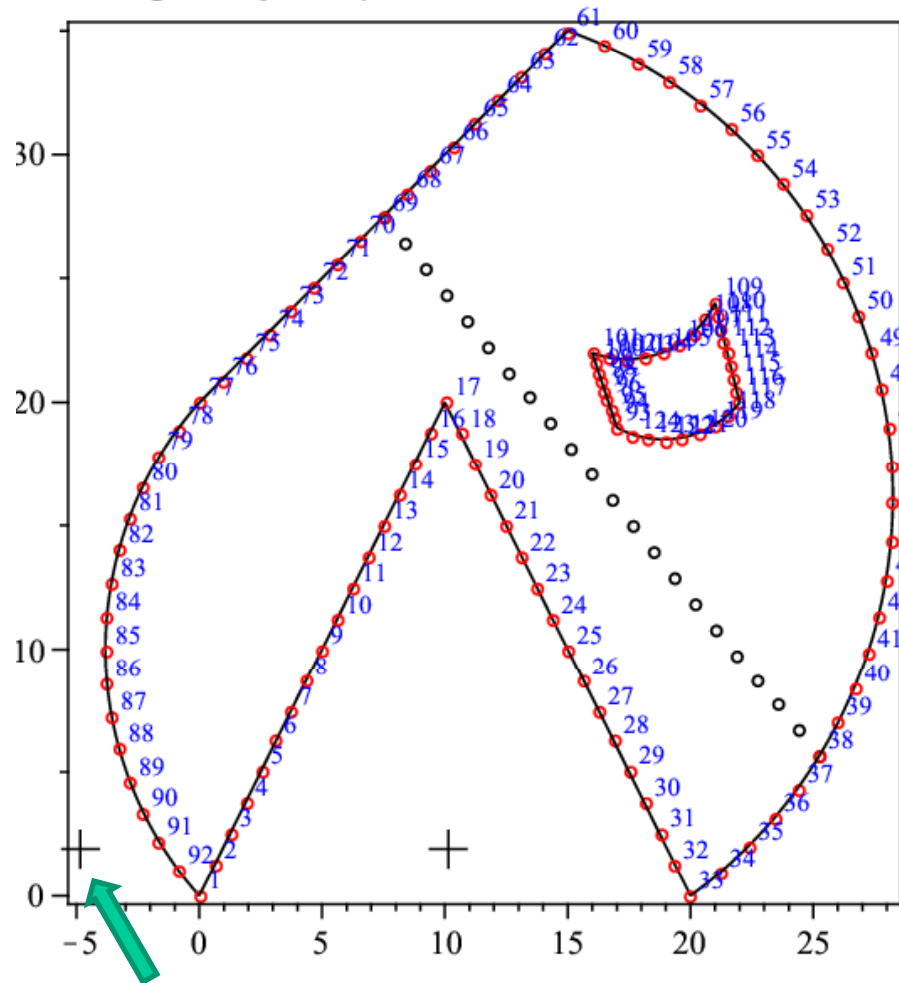
$$\text{sol\_9: } u(x, y) = \frac{\ln((x - x_1)^2 + (y - y_1)^2)}{4\pi}$$



# An expedite formulation of the BEM

## Some numerical assessments for 2D potential problems

*Irregularly shaped structure with 124 nodes*



# Contents of this presentation

- Variationally-based, hybrid boundary and finite elements



- Developments for time-dependent problems
- Developments in gradient elasticity
- Dislocation-based formulations (for fracture mechanics)
- From the collocation (conventional of hybrid) boundary element method to a meshless formulation  
Or: **The expedite boundary element method**



# GENERALIZED HELLINGER-REISSNER POTENTIAL

$$\int_{t_0}^{t_1} \left( - \int_{\Omega} (\delta \sigma_{ij}, j - \rho \delta \ddot{u}_i) (u_i - \tilde{u}_i) d\Omega + \int_{\Gamma} \delta \sigma_{ij} \eta_j (u_i - \tilde{u}_i) d\Gamma + \right. \\ \left. + \int_{\Omega} \delta \tilde{u}_i (\sigma_{ij}, j + \bar{f}_i - \rho \ddot{u}_i) d\Omega - \int_{\Gamma} \delta \tilde{u}_i (\sigma_{ij} \eta_j - \bar{t}_i) d\Gamma \right) dt = 0$$

$$\delta \tilde{u}_i = 0 \quad \text{along} \quad \Gamma_u$$

$$\delta u_i = 0 \quad \text{at both time interval extremities } t_0 \text{ and } t_1$$

# Literature Review – part I (continued)

Przemieniecki, J.S. (1968). *Theory of Matrix Structural Analysis*,  
Dover Publications, New York  
(displacement-based free vibration analysis – truss and beam):

$$\mathbf{K} = \mathbf{K}_0 - \omega^2 \mathbf{M}_0 - \omega^4 (\mathbf{M}_2 - \mathbf{K}_4) - \omega^6 (\mathbf{M}_4 - \mathbf{K}_6) + O(\omega^8)$$

Voss, H., 1987, A New Justification of Finite Dynamic Element Methods, *Numerical Treatment of Eigenvalue Problems*, Vol. 4, 232-242, eds. J. Albrecht, L. Collatz, W. Velte, W. Wunderlich, Int. Series on Num. Maths. 83, Birkhäuser Verlag, Stuttgart

Gupta, Paz: several other displacement-based elements for plane-state problems. Coined *finite dynamic elements*, although they only have been applied to free vibration analysis (better, then: *finite harmonic elements!*)

$$\mathbf{K} = \mathbf{K}_0 - \omega^2 (\mathbf{M}_0 - \mathbf{K}_2) - \omega^4 (\mathbf{M}_2 - \mathbf{K}_4) - \omega^6 (\mathbf{M}_4 - \mathbf{K}_6) + O(\omega^8)$$

Directly based on Pian and Przemieniecki: **General , consistent finite and boundary dynamic element families: acoustics, free vibration, transient problems via an advanced modal analysis.**

Dumont, N. A. & Oliveira, R., (1993) 1997. The exact dynamic formulation of the hybrid boundary element method, *Procs. XVIII CILAMCE*, Brasília, 29 - 31 October, 357-364

Dumont, N. A. & Oliveira, R., 2001. From frequency-dependent mass and stiffness matrices to the dynamic response of elastic systems. *Int. J. Sol. Struct.*, **38**, 10-13, 1813-1830

# Literature Review – part II

Lancaster, P., 2002. *Lambda-Matrices & Vibrating Systems*, Dover Publications  
(from 1966)

$$(\mathbf{F}_0 + \omega^2 \mathbf{F}_1 + \omega^4 \mathbf{F}_2 + \dots) \mathbf{p}^* = (\mathbf{H}_0 + \omega^2 \mathbf{H}_1 + \omega^4 \mathbf{H}_2 + \dots) \mathbf{d}$$

$$(\mathbf{H}_0^T + \omega^2 \mathbf{H}_1^T + \omega^4 \mathbf{H}_2^T + \dots) \mathbf{p}^* = \mathbf{p}$$

Dumont, N. A.: "On the Inverse of Generalized Lambda Matrices with Singular Leading Term", *IJNME*, Vol 66(4), 571-603, 2006

$$(\mathbf{K}_0 - \omega^2 \mathbf{M}_1 - \omega^4 \mathbf{M}_2 + \dots) \mathbf{d} = \mathbf{p}(\omega)$$

(Przemieniecki)

$$\left\{ \begin{array}{l} \mathbf{H}^T \mathbf{V} = \mathbf{0} \\ \mathbf{F}_0 \mathbf{V} = \mathbf{0} \\ \mathbf{H}^T \mathbf{F}^{-1} \mathbf{H} = \mathbf{K} \end{array} \right.$$

## Nonlinear eigenvalue problem:

Dumont, N. A.: "On the solution of generalized non-linear complex-symmetric eigenvalue problems", *IJNME*, Vol 71, 1534-1568, 2007

- Sleipjen, G. L. G., Van der Horst, H. A., 1996, A Jacobi-Davidson iteration method for linear eigenvalue problems, *SIAM J. Matrix Anal. Appl.*, 17, 401-425
- Arbenz, P, Hochstenbach, M. E., 2004. A Jacobi-Davidson method for solving complex-symmetric eigenvalue problems, *SIAM J. Sci. Comput.* 25, 5, 1655-1673

# Problems with viscous damping

Nonlinear eigenproblem:

$$\mathbf{K}_0 \Phi - \sum_{j=1}^n \left( i\mathbf{C}_j \Phi \Omega^{2j-1} + \mathbf{M}_j \Phi \Omega^{2j} \right) = \mathbf{0}$$

Augmented formulation:

$$\begin{bmatrix} \mathbf{K}_0 & \mathbf{0} & \mathbf{0} & \mathbf{0} & \cdots & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_1 & i\mathbf{C}_2 & \mathbf{M}_2 & \cdots & \mathbf{M}_n \\ \mathbf{0} & i\mathbf{C}_2 & \mathbf{M}_2 & i\mathbf{C}_3 & \cdots & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_2 & i\mathbf{C}_3 & \ddots & \cdots & \mathbf{0} \\ \vdots & \vdots & \vdots & \vdots & \ddots & \vdots \\ \mathbf{0} & \mathbf{M}_n & \mathbf{0} & \mathbf{0} & \cdots & \mathbf{0} \end{bmatrix} \begin{bmatrix} \Phi_{00} & \Phi_{01} & \cdots & \Phi_{0,n-1} \\ \Phi_{10} & \Phi_{11} & \cdots & \Phi_{1,n-1} \\ \vdots & \vdots & \ddots & \vdots \\ \Phi_{n-1,0} & \Phi_{n-1,1} & \cdots & \Phi_{n-1,n-1} \end{bmatrix} - \begin{bmatrix} i\mathbf{C}_1 & \mathbf{M}_1 & i\mathbf{C}_2 & \mathbf{M}_2 & \cdots & \mathbf{M}_n \\ \mathbf{M}_1 & i\mathbf{C}_2 & \mathbf{M}_2 & i\mathbf{C}_3 & \cdots & \mathbf{0} \\ i\mathbf{C}_2 & \mathbf{M}_2 & i\mathbf{C}_3 & \cdots & \cdots & \mathbf{0} \\ \mathbf{M}_2 & i\mathbf{C}_3 & \vdots & \ddots & \cdots & \mathbf{0} \\ \vdots & \vdots & \vdots & \vdots & \ddots & \vdots \\ \mathbf{M}_n & \mathbf{0} & \mathbf{0} & \mathbf{0} & \cdots & \mathbf{0} \end{bmatrix} \begin{bmatrix} \Phi_{00} & \Phi_{01} & \cdots & \Phi_{0,n-1} \\ \Phi_{10} & \Phi_{11} & \cdots & \Phi_{1,n-1} \\ \vdots & \vdots & \ddots & \vdots \\ \Phi_{n-1,0} & \Phi_{n-1,1} & \cdots & \Phi_{n-1,n-1} \end{bmatrix} \begin{bmatrix} \Omega_0 & \mathbf{0} & \cdots & \mathbf{0} \\ \mathbf{0} & \Omega_1 & \cdots & \mathbf{0} \\ \vdots & \vdots & \ddots & \vdots \\ \mathbf{0} & \mathbf{0} & \cdots & \Omega_{n-1} \end{bmatrix} = \mathbf{0}$$

Orthogonality properties:

$$\sum_{j=1}^n \left( \sum_{k=2}^{2j} \Omega^{k-2} \Phi^T i\mathbf{C}_j \Phi \Omega^{2j-k} + \sum_{k=1}^{2j} \Omega^{k-1} \Phi^T \mathbf{M}_j \Phi \Omega^{2j-k} \right) = \mathbf{I}$$

$$\Phi^T \mathbf{K}_0 \Phi + \sum_{j=1}^n \left( \sum_{k=1}^{2j-2} \Omega^k \Phi^T i\mathbf{C}_j \Phi \Omega^{2j-k-1} + \sum_{k=1}^{2j-1} \Omega^k \Phi^T \mathbf{M}_j \Phi \Omega^{2j-k} \right) = \Omega$$

# Summary

Equation in the time domain

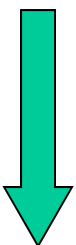
$$\mathbf{K}_0 \mathbf{d} - \sum_{i=1}^n (-1)^i \mathbf{M}_i \frac{\partial^{2i} \mathbf{d}}{\partial t^{2i}} = \mathbf{p}(t) \quad (4)$$

e. g.

$$\left( \mathbf{K}_0 \mathbf{d} + \mathbf{M}_1 \frac{\partial^2 \mathbf{d}}{\partial t^2} - \mathbf{M}_2 \frac{\partial^4 \mathbf{d}}{\partial t^4} + \mathbf{M}_3 \frac{\partial^6 \mathbf{d}}{\partial t^6} = \mathbf{p}(t) \right)$$

(Set of coupled, higher-order differential equations)

Mode superposition :


$$\left[ \text{from } (\mathbf{K}(\omega) - \omega^2 \mathbf{M}(\omega)) \boldsymbol{\varphi} = \mathbf{0} \right]$$
$$\mathbf{d} = \Phi \boldsymbol{\eta} \quad (15)$$

$$\Omega^2 \boldsymbol{\eta} + \ddot{\boldsymbol{\eta}} = \Phi^T \mathbf{p} \quad (17)$$

(Set of uncoupled, second-order differential equations)

# Frequency-domain formulation

$$\sigma_{ji,j} + b_i + \rho k^2 u_i = 0 \quad \text{where} \quad k^2 = \omega^2 + 2i\zeta\omega$$

**Solution:**

(Laplace/Fourier transforms or)

**Advanced** mode-superposition technique

Structural dynamics

$$\Omega^2 (\boldsymbol{\eta} - \boldsymbol{\eta}^b) + \ddot{\boldsymbol{\eta}} - \ddot{\boldsymbol{\eta}}^b = \boldsymbol{\Phi}^T (\mathbf{p} - \mathbf{p}^b)$$

Diffusion-type problems

$$\Omega (\boldsymbol{\eta} - \boldsymbol{\eta}^b) + \dot{\boldsymbol{\eta}} - \dot{\boldsymbol{\eta}}^b = \boldsymbol{\Phi}^T (\mathbf{p} - \mathbf{p}^b)$$

Structural dynamics with viscous damping

$$\Omega (\boldsymbol{\eta} - \boldsymbol{\eta}^b) - i (\dot{\boldsymbol{\eta}} - \dot{\boldsymbol{\eta}}^b) = \boldsymbol{\Phi}^T (\mathbf{p} - \mathbf{p}^b)$$

# Initial displacements and velocities

$$\left\{ \begin{array}{l} \boldsymbol{\eta}_{el} = \left[ \boldsymbol{\Phi}_{el}^T \mathbf{K}_0 \boldsymbol{\Phi}_{el} \right]^{-1} \boldsymbol{\Phi}_{el}^T \mathbf{K}_0 \mathbf{d} \\ \boldsymbol{\eta}_{rig} = \boldsymbol{\Phi}_{rig}^T \mathbf{M}_1 \mathbf{d} \end{array} \right.$$

In case of viscous damping:


$$\left\{ \begin{array}{l} \mathbf{d} \\ i\dot{\mathbf{d}} \end{array} \right\} = \left[ \begin{array}{cc} \boldsymbol{\Phi}_1 & \boldsymbol{\Phi}_2 \\ \boldsymbol{\Phi}_1 \boldsymbol{\Omega}_1 & \boldsymbol{\Phi}_2 \boldsymbol{\Omega}_2 \end{array} \right] \left\{ \begin{array}{l} \boldsymbol{\eta}_1 \\ \boldsymbol{\eta}_2 \end{array} \right\}$$
$$\left[ \begin{array}{cc} \boldsymbol{\Phi}_1^T \mathbf{K}_0 \boldsymbol{\Phi}_1 & \boldsymbol{\Phi}_1^T \mathbf{K}_0 \boldsymbol{\Phi}_2 \\ \boldsymbol{\Phi}_1^T \mathbf{K}_0 \boldsymbol{\Phi}_1 \boldsymbol{\Omega}_1 & \boldsymbol{\Phi}_1^T \mathbf{K}_0 \boldsymbol{\Phi}_2 \boldsymbol{\Omega}_2 \end{array} \right] \left\{ \begin{array}{l} \boldsymbol{\eta}_1 \\ \boldsymbol{\eta}_2 \end{array} \right\} = \left\{ \begin{array}{l} \boldsymbol{\Phi}_1^T \mathbf{K}_0 \mathbf{d} \\ i\boldsymbol{\Phi}_1^T \mathbf{K}_0 \dot{\mathbf{d}} \end{array} \right\}$$

# EVALUATION OF RESULTS AT INTERNAL POINTS

$$\mathbf{p}^*(\omega) = \mathbf{S}(\omega) [\mathbf{d}(\omega) - \mathbf{d}^b(\omega)] \quad \text{in which} \quad \mathbf{S}(\omega) = \mathbf{F}^{-1}(\omega) \mathbf{H}(\omega)$$



replaced by

$$\mathbf{p}^*(\omega) \approx \sum_{i=0}^n \omega^{2i} \mathbf{S}_i [\mathbf{d}(\omega) - \mathbf{d}^b(\omega)] \quad \text{with} \quad \sum_{i=0}^n \omega^{2i} \mathbf{S}_i = \left( \sum_{i=0}^n \omega^{2i} \mathbf{F}_i \right)^{-1} \sum_{i=0}^n \omega^{2i} \mathbf{H}_i$$



$$\mathbf{p}^*(t) = \sum_{i=0}^n \mathbf{S}_i \Phi \Omega^{2i} (\boldsymbol{\eta} - \boldsymbol{\eta}^b)$$



# EVALUATION OF RESULTS AT INTERNAL POINTS

$$\mathbf{p}^*(t) = \sum_{i=0}^n \mathbf{S}_i \Phi \Omega^{2i} (\eta - \eta^b)$$

$$\mathbf{u}(t) = \sum_{j=0}^m \sum_{i=0}^n \omega_j^{2i} \mathbf{u}_i^* \mathbf{p}^* \equiv \sum_{i=0}^n \mathbf{u}_i^* \Omega^{2i} \mathbf{p}^*$$



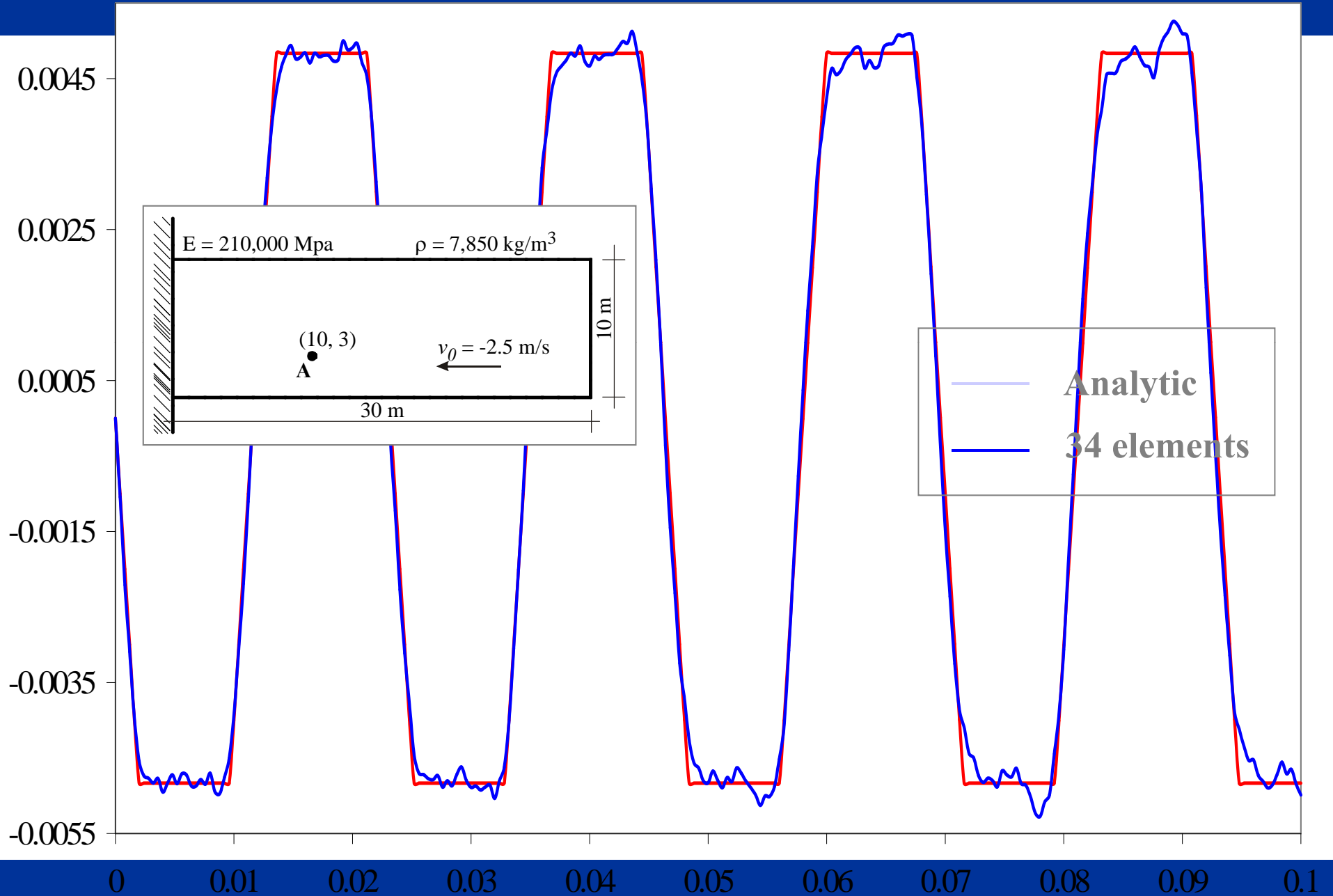
$$\mathbf{u}(t) = \sum_{i=0}^n \sum_{j=0}^i \mathbf{u}_j^* \mathbf{S}_{i-j} \Phi \Omega^{2i} (\eta - \eta^b)$$

for  $n = 3$

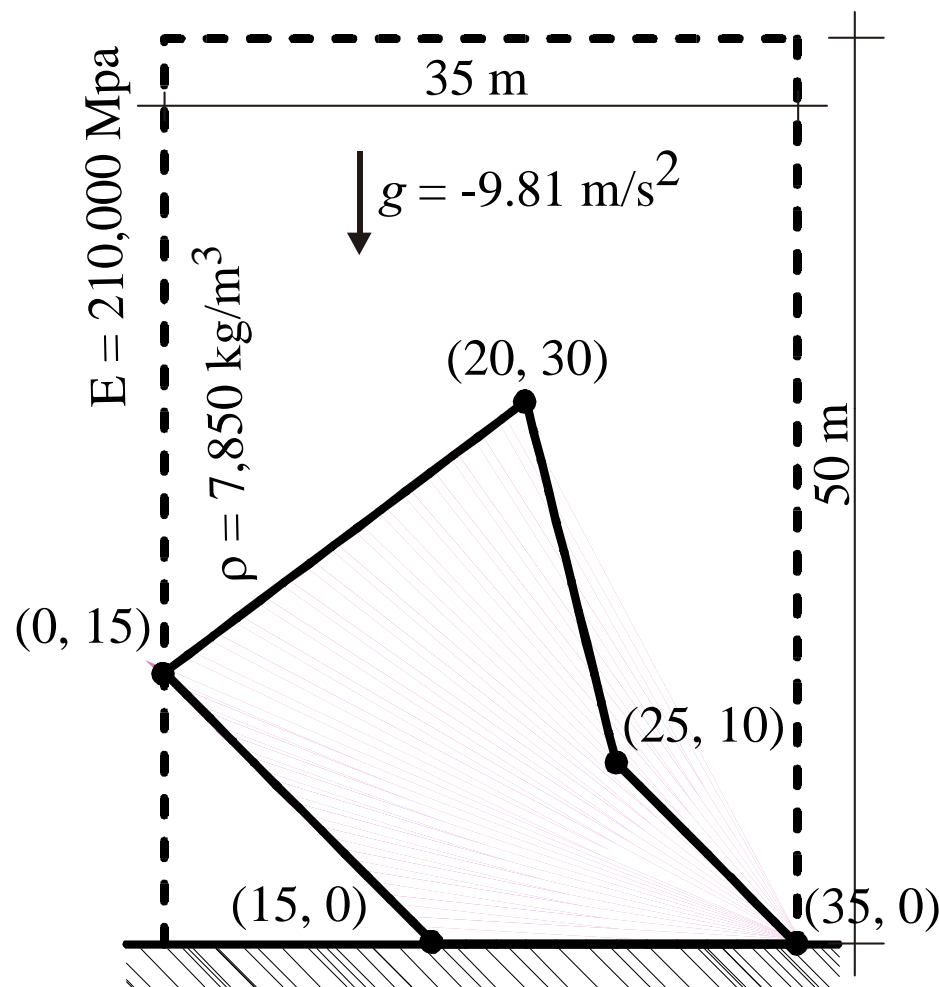
$$\begin{aligned} \mathbf{u}(t) = & \left[ \mathbf{u}_0^* \mathbf{S}_0 \Phi + (\mathbf{u}_0^* \mathbf{S}_1 + \mathbf{u}_1^* \mathbf{S}_0) \Phi \Omega^2 + (\mathbf{u}_0^* \mathbf{S}_2 + \mathbf{u}_1^* \mathbf{S}_1 + \mathbf{u}_2^* \mathbf{S}_0) \Phi \Omega^4 + \right. \\ & \left. + (\mathbf{u}_0^* \mathbf{S}_3 + \mathbf{u}_1^* \mathbf{S}_2 + \mathbf{u}_2^* \mathbf{S}_1 + \mathbf{u}_3^* \mathbf{S}_0) \Phi \Omega^6 \right] \eta \end{aligned}$$

$$\left( \mathbf{S} \text{ is required for } \mathbf{p}^* = \mathbf{F}^{-1} \mathbf{H} \mathbf{d} \equiv \mathbf{S} \mathbf{d} \right)$$

# Example 1: Initial velocity – Displacement at point A



# Transient Analysis – Example 2: Gravitational force



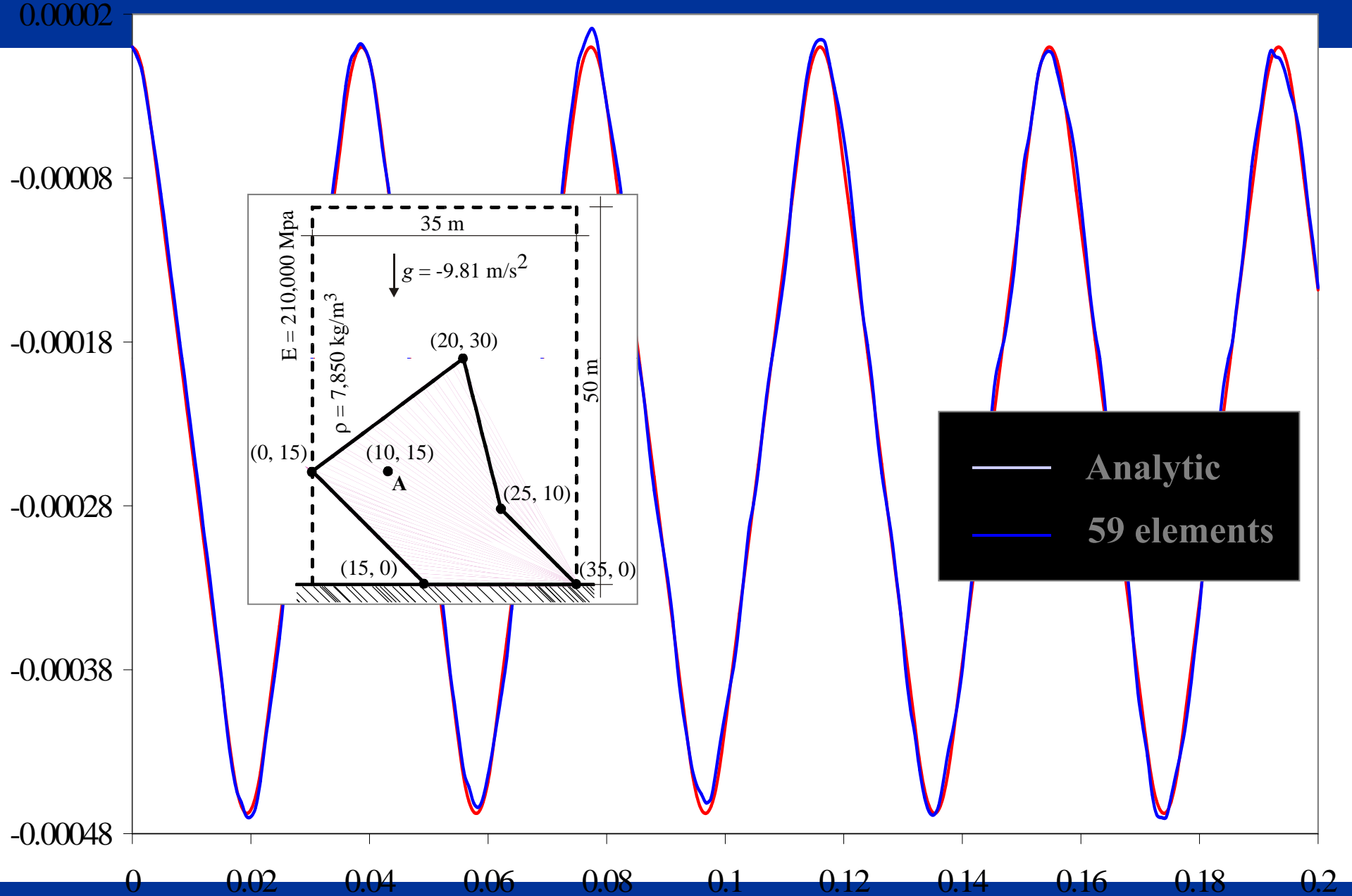
59 equally spaced,  
linear boundary elements

$$u = \sum_{n=1}^{\infty} \frac{4g \sin(k_n x) [1 - \cos(\omega_n t)]}{(2n-1) \pi \omega_n^2}$$

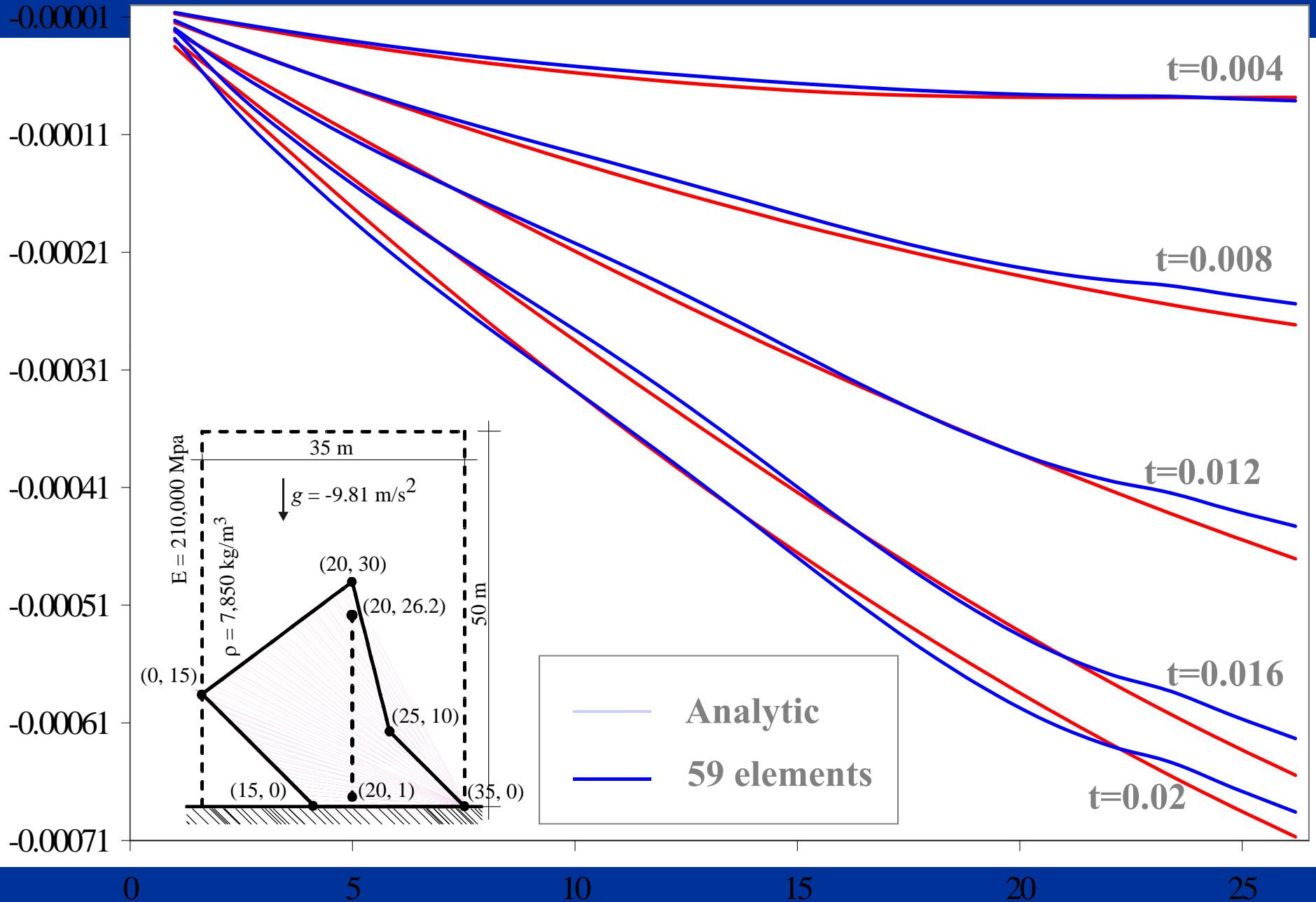
$$\left\{ \begin{array}{l} \omega_n = k_n \sqrt{E/\rho} \\ k_n = \frac{(2n-1)\pi}{2L} \end{array} \right.$$

$v = 0$  (can be solved in the frame of the theory of potential)

## Example 2: Gravitational force – Displacement at point A

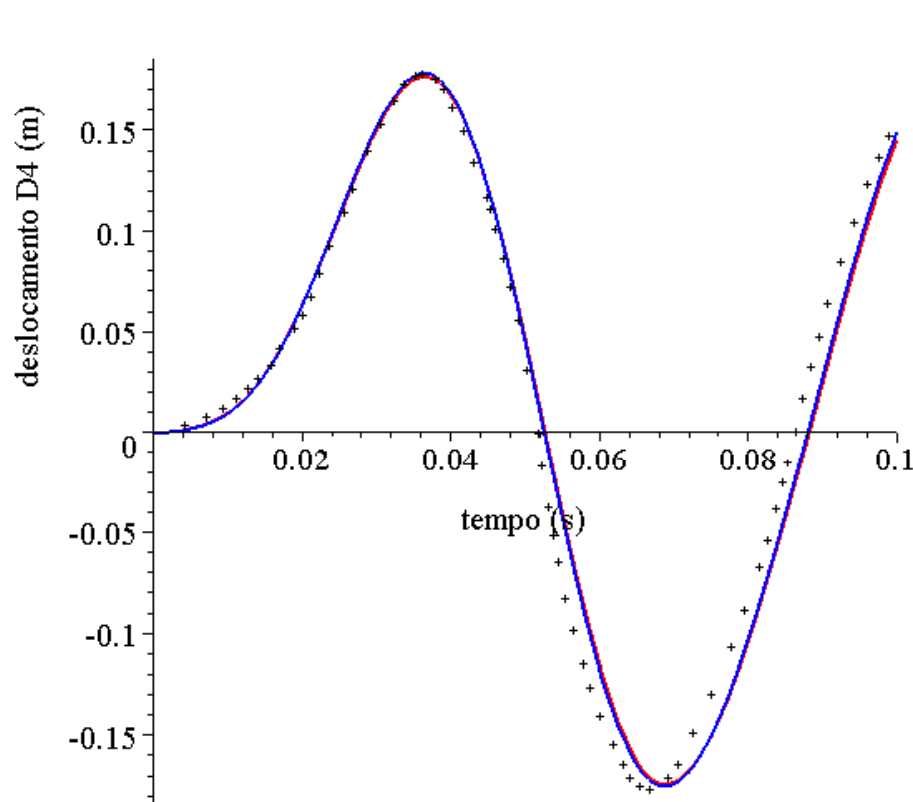


## Example 2: Gravitational force – Displacements along a line segment

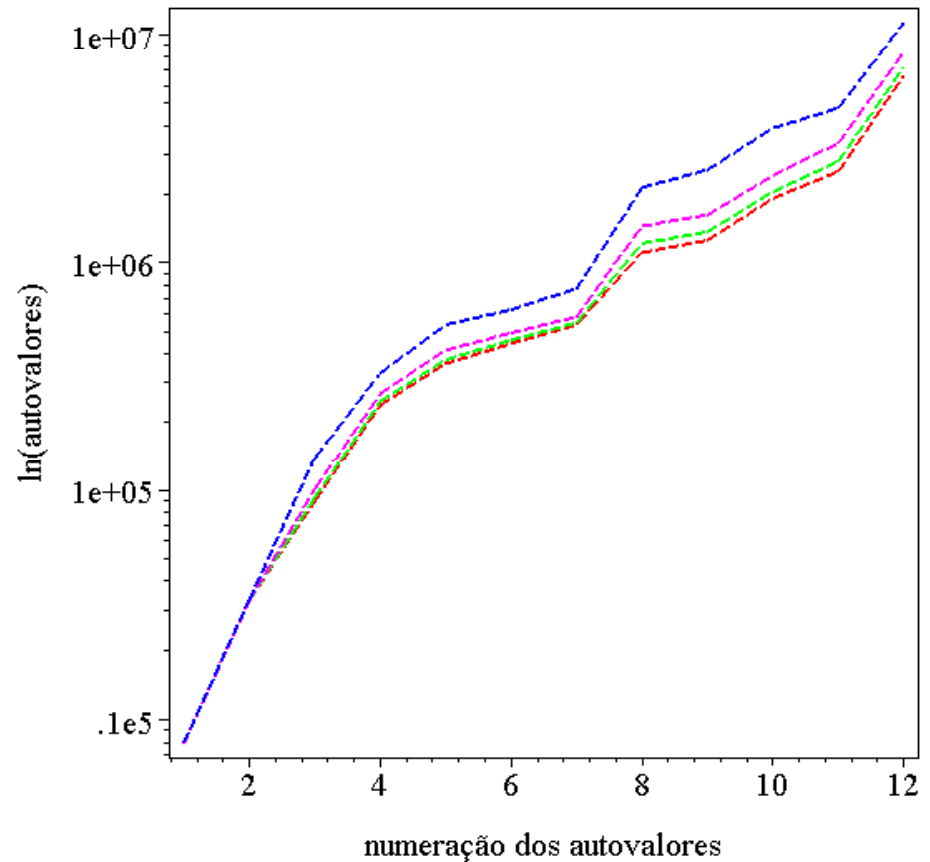




# Plane frame with 6 members and 12 d.o.f.



Displacement results with 1 and 4 mass matrices, as compared with values given by Weaver and Johnston.



Eigenvalues obtained for the problem with 1, 2, 3 and 4 mass matrices.

# Contents of this presentation

- Variationally-based, hybrid boundary and finite elements
- Developments for time-dependent problems
- • Developments in gradient elasticity
- Dislocation-based formulations (for fracture mechanics)
- From the collocation (conventional or hybrid) boundary element method to a meshless formulation  
Or: **The expedite boundary element method**



# A Family of 2D and 3D Hybrid Finite Elements for Strain Gradient Elasticity

Ney Augusto Dumont  
Daniel Huamán Mosqueira  
e-mail: dumont@puc-rio.br



**PONTIFÍCIA UNIVERSIDADE CATÓLICA  
DO RIO DE JANEIRO**



# Conclusions <sup>(1)</sup>

$$\sigma_{ji} = \tau_{ji} - g^2 \tau_{ji,kk} \quad \text{Total stress}$$

- The hybrid BEM / FEM (a two-field formulation) is a natural variational tool to deal with gradient elasticity
- Singular and nonsingular fundamental solutions were either redeveloped for BEM or originally developed for FEM gradient elasticity implementations
- General families of finite elements were obtained
- Extension to time-dependent problems in the frequency domain is straightforward (in progress – already done for truss and beam elements)
- Simple implementations for truss and beam elements: was a fruitful apprenticeship (disagreement with some results in the literature - symmetry and representation of constant strain state)
- Treatment of the normal displacement gradient on  $\Gamma$  is different from Mindlin's proposition (our results still remain to be validated)
- Numerical examples for 2D problems are being implemented

# Conclusions (2)

$$\sigma_{ji} = \tau_{ji} - g^2 \tau_{ji,kk} \quad \text{Total stress}$$

- .
- .
- .
- **Treatment of the normal displacement gradient on  $\Gamma$  is different from Mindlin's proposition (our results still remain to be validated)**
- Numerical examples for 2D problems are being implemented:
  - Orthogonality to rigid body displacements: OK!
  - Symmetry of the flexibility matrix  $\mathbf{F}^*$ : OK!
  - Patch tests for linear displacements fields: OK!
  - Patch tests for non-linear displacement fields: convergence still questionable.

# Contents of this presentation

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Or: **The expedite boundary element method**

# Conclusions

## Expedite formulation of the boundary element method (in progress)

- **Implementation extremely advantageous for:**
  - **large problems**
  - **problems with complicated fundamental solutions (time-dependent, axisymmetric, for gradient elasticity, etc)**
- **Numerical accuracy still under investigation, but comparable to the conventional BEM**
- **Evaluation of results at internal points requires no further integrations**
- **However, results close to the boundary require the knowledge of the null space  $V$  (which may be obtained via Gauss-Seidel iteration in principle with a simple pre-conditioning of  $H^T$ )**

## Work in progress

- **Implementation in Fortran for large 2D potential and elasticity problems in the frequency domain**
- **Implementation for strain gradient elasticity**